# Evaluation of the Stress and Displacement Behavior of Different LS-Dyna Element Types in Combination with Different Anti-Hourglassing Formulations and Initial Element Deformations 

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## Outline

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5.1 Results for displacement-based loading
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Summary

## 1. Motivation



Which solid element type and which anti-hourglassing formulation is suitable for thin-walled structures?


Solid ELFORM 1, 2, -1, -2, ... T-Shell ELFORM 1
Shell ELFORM 1, 2, 16, ...


What happens for initially deformed elements?

## 2. Elements under consideration

## Solid elements

ELFORM 1: - Underintegrated constant stress element (standard solid element)

- Fastest element in this test

ELFORM 2: - Fully integrated element (tendency for locking)

- Slower than ELFORM 1

ELFORM -1: - Similar to ELFORM 2, but accounted for poor aspect ratios on order to reduce shear locking

- Slower than ELFORM 2

ELFORM -2: - Similar to ELFORM 2, but accounted for poor aspect ratios on order to reduce shear locking

- Higher costs than for ELFORM -1 because of more accurate formulation


## T-Shell elements

ELFORM 1: - Underintegrated element

- Appears to the user as 8-node brick element (but plane stress is assumed)


## Shell elements

ELFORM 1: - Underintegrated element (Hughes-Liu formulation)
ELFORM 2: - Underintegrated element (Belytschko-Tsai formulation, standard shell element)
ELFORM 16: - Fully integrated element

- Higher computational costs than ELFORM 1 and 2
- Preferred for implicit calculations

For more details please refer to: Hallquist, J. O.: "LS-DYNA Theory Manual", Livermore Software Technology Corporation, 2006

## 3. Model and boundary conditions

Solid model


## Boundary conditions:

- Node 1 and node 2 are fixed in y-direction, node 2 and node 3 are fixed in x-direction
- Lower cylinders are fixed, upper cylinder is loaded


## Loading:

- Upper cylinder is loaded by a force of 0.1 N in negative z-direction or with a prescribed displacement of 0.5 mm in negative z-direction resp.


## 3. Model and boundary conditions

## Load curve:



Time integration: All computations are done with explicit time integration (mpp solver)
Damping: Global damping of $10 \mathrm{e} 4 \mathrm{~s}^{-1}$ is applied (to achieve quasi-static solution)
Initial element deformations: All computations are done without and with initial element deformations:


Undeformed element


Initially deformed element


To achieve initial element deformations, the taper angle $\alpha$ of the middle elements is varied between $\alpha=0^{\circ}, \alpha=5^{\circ}$, $\alpha=10^{\circ}$ and $\alpha=20^{\circ}$.

Hourglass Control: All computations are done with HG 4, 5 and 6 with default control coefficients

## 4. Test results with undeformed elements

4.1 Max. bending stresses for DISPLACEMENT-BASED loading and different anti-hourglassing formulations



- Influence of hourglass control onto computed stresses only for solid ELFORM 1
- Shell elements compute more accurate results due to position of integration point


## 4. Test results with undeformed elements


4.1 Max. bending stresses for DISPLACEMENT-BASED loading and different anti-hourglassing formulations


## Solid ELFORM 1:

- HG 4 and HG 5 cause an increase of the stiffness which results in higher stresses (especially in y-direction)
- This specific reaction can also be observed for the internal energy functions
- The decrease of the internal energy of the model without HG control results from an increase of the hourglass energy
- Without hourglassing, model with no HG control would lead to the same result like model with HG control 6


## 4. Test results with undeformed elements



## 5. Test results with initially deformed elements



Most meshes of real parts do not contain only perfectly brick-shaped elements, where all edges are perpendicular to each other.


How do such initially deformed solid elements behave in the 3 -point-bending test?


Here, variation of so-called taper angle:


## 5. Test results with initially deformed elements

### 5.1 Max. bending stresses for DISPLACEMENT-BASED loading



## 5. Test results with initially d

5.1 Max. bending stresses for DISPLACEMENT-BA


change between tensile and compressive bending stresses

| ¢ | Solid -2 | Solid-1 Solid 1 Solid 2 $\quad$ T-Shell 1 |
| :---: | :---: | :---: |
| 㐫 -100.0 |  |  |

## 5. Test results with initially deformed elements

### 5.1 Max. bending stresses for DISPLACEMENT-BASED loading



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## 5. Test results with initially deformed elements

### 5.1 Max. bending stresses for DISPLACEMENT-BASED loading



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## 5. Test results with initially deformed elements

### 5.1 Max. bending stresses for DISPLACEMENT-BASED loading





## 5. Test results with initially deformed elements

### 5.2 Max. displacements for FORCE-BASED loading





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## 5. Test results with initially deformed elements

5.2 Max. displacements for FORCE-BASED loading


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## 5. Test results with initially deformed elements

### 5.2 Max. displacements for FORCE-BASED loading



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## 5. Test results with initially deformed elements

5.2 Max. displacements for FORCE-BASED loading


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## 5. Test results with initially deformed elements

### 5.2 Max. displacements for FORCE-BASED loading



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## 5. Test results with initially deformed elements

### 5.2 Max. displacements for FORCE-BASED loading



## 6. Comparison of computation time

- The table given below shows the total CPU times [sec] (elapsed time of d3hsp files) for all force-based calculations with a taper angle of $0^{\circ} \mathrm{deg}$ and hourglass controls 5 or 6
- Because HG6 is not implemented for shell elements, they are not listed in the second table.
- All CPU times are normalised with respect to solid ELFORM 1 (HG6).
- As expected, shell elements are much faster than solid elements (less DOF, bigger time-step size).
- Solid ELFORM 1 is the fastest solid element but ELFORM -1 is only about $30 \%$ slower
- There is nearly no difference in the computation time between hourglass control 5 and 6 in LS Dyna.

| HG5: |  | Total CPU time [s] | Total CPU time - normalised |
| :---: | :---: | :---: | :---: |
| Solid | ELFORM -2 | 1917.700000 | 276.1 |
|  | ELFORM -1 | 913.280000 | 131.5 |
|  | ELFORM 1 | 677.950000 | 97.6 |
|  | ELFORM 2 | 845.120000 | 121.7 |
| T-Shell | ELFORM 1 | 193.450000 | 27.8 |
| Shell | ELFORM 1 | 11.157000 | 1.6 |
|  | ELFORM 2 | 11.156000 | 1.6 |
|  | ELFORM 16 | 9.031000 | 1.3 |
| HG6: |  |  |  |
|  | ELFORM -2 | 1920.200000 | Total CPU time [s] |
|  | ELotal CPU time - normalised |  |  |
|  | ELFORM 1 1 | 915.170000 | 276.5 |
|  | ELFORM 2 | 843.550000 | 131.8 |
| T-Shell | ELFORM 1 | 192.830000 | 100 |
|  |  |  | 121.5 |



## Summary

## Initially undeformed elements



Displacement-based computations:

- Shell elements compute most accurate stress results due to position of integration points
- Stress results of solid ELFORM 1 strongly depend on the used hourglass control algorithm

Force-based computations:

- Only HG control 6 leads to good results in terms of stresses and displacements for solid ELFORM 1
- HG control 4 and 5 lead to too stiff structures and too low stresses for solid ELFORM 1

Initially deformed elements (taper angle $>0^{\circ}$ )


Displacement-based computations:

- Results of solid ELFORM 1 are highly sensitive with respect to the taper angle
- For taper angles $>0^{\circ}$ Solid ELFORM 1 leads to good results only in combination with HG 5
- Solid ELFORM -1 leads to very good results for all taper angles

Forced-based computations:

- Solid ELFORM 1 reacts to stiff in combination with HG control 4 or 5 (good results in combination with HG 6 but only for taper angle $=0^{\circ}$ )
- Solid -1 leads to good results for all taper angles in terms of stresses and displacements
- All test have also been carried out with negative taper angles and led to quantitatively similar results



## Thank you for your attention!

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