# Numerical simulation of Dynamic Failure: ballistic tests on Titanium-6Al-4V

Paul Du Bois, Sean Haight, Leyu Wang, Steve Kan,
Bill Emmerling,
Kelly Carney, Mike Pereira, Amos Gilat, Jeremy Seidt







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### **Introduction**

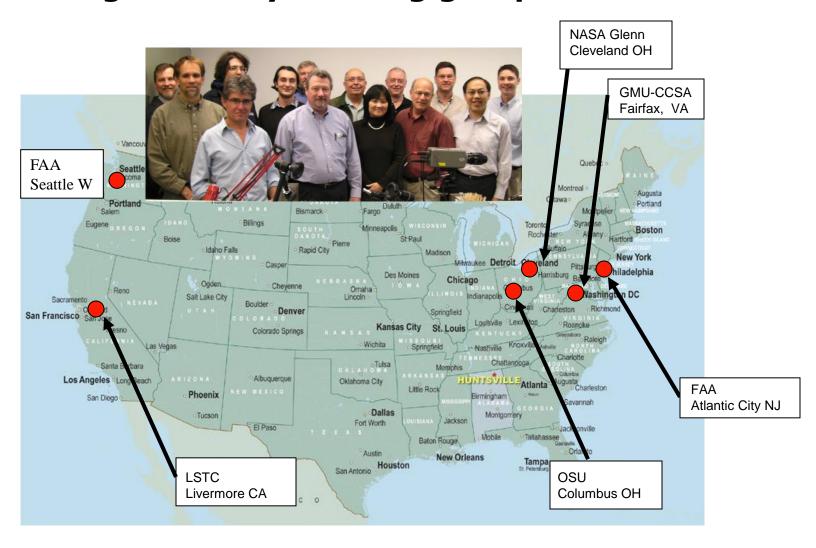


- Part of a research program conducted by FAA William J Hughes Technical Center (NJ)
- material testing performed by OSU
- ballistic testing performed by NASA/GRC
- numerical simulations performed by GMU-CCSA
- involved the implementation in LS-DYNA of a tabulated generalisation of the Johnson-Cook material law with regularisation to accommodate simulation of failure in ductile materials: MAT\_224 or MAT\_TABULATED\_JOHNSON\_COOK



- previously published results in :
- •A Generalized, Three Dimensional Definition, Description and Derived Limits of the Triaxial Failure of Metals, Carney, DuBois, Buyuk, Kan, Earth&Sky, march 2008

### FAA engine safety working group

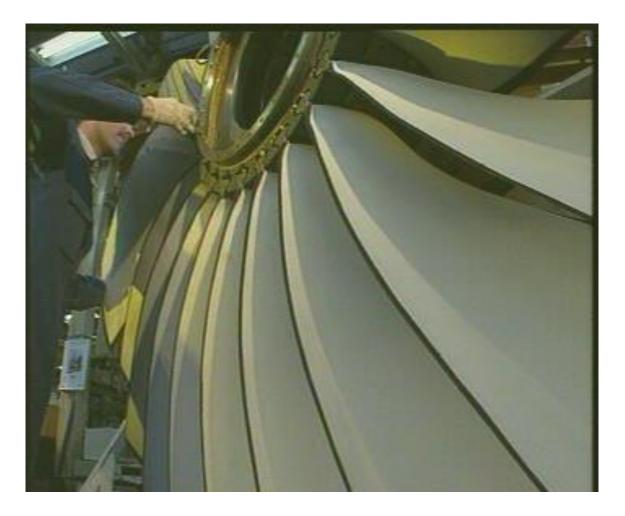


### **Background: blade-out events**

- Aircraft Safety depends upon sound engine containment designs, and upon realistic evaluation of the damage from uncontained engine debris.
- The program addresses the modeling of impact between the blades and case, or between the fragments and non-engine aircraft structure
- The program has developed an extensive material test database and has modeled many different tests to evaluate the overall applicability of a single material model to the larger overall problem





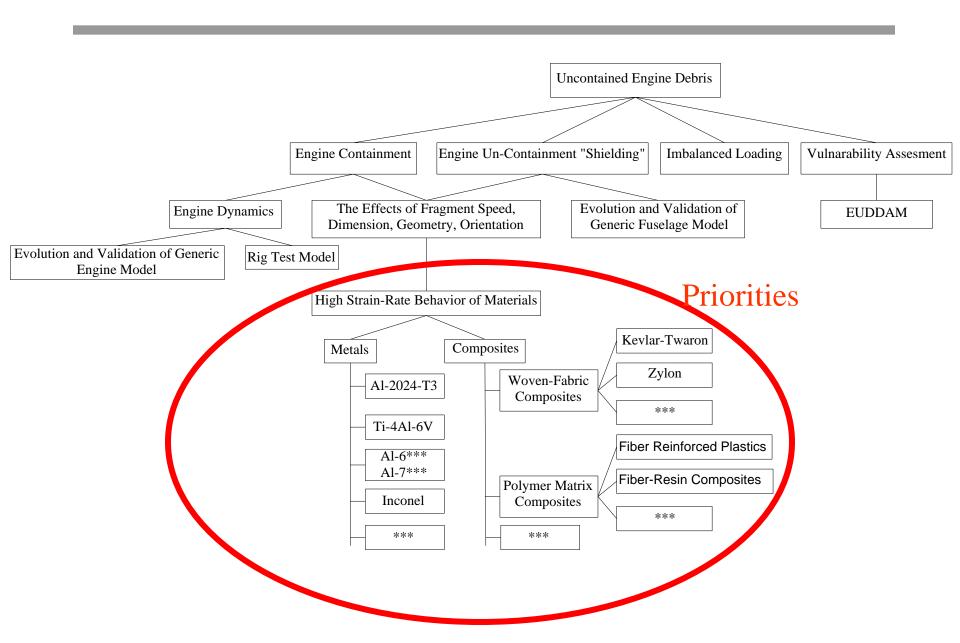




Fan blades of Trent

Mandatory full scale engine containment test

#### **Numerical Simulation of Dynamic Failure: ballistic tests on Titanium-6Al-4V**



### References

- Ravi Shriram Yatnalkar, Experimental Investigation of Plastic Deformation of Ti-6Al-4V under Various Loading Conditions, B.E, Thesis, The Ohio State University, 2010
- Jeremy Daniel Seidt, Plastic Deformation and Ductile Fracture of 2024-T351 Aluminum under Various Loading Conditions, Dissertation, The Ohio State University, 2010
- Jeremiah Thomas Hammer, Plastic Deformation and Ductile fracture of Ti-6Al-4V under Various loading Conditions, Dissertation, The Ohio State University, 2012
- Murat Buyuk, Development of a tabulated thermo-viscoplastic material model with regularized failure for dynamic ductile failure prediction of structures under impact loading, Dissertation, The George Washington University, 2013

### **Development of MAT\_224 in LS-DYNA**

- started development in november 2006
- production version available in Is971-R4.2
- current presentation is based on implementation in Is971-R6.1
- developed on the basis of MAT\_024 with VP=1
- available for fully and underintegrated shell and solid elements
- full keyword code : \*MAT\_TABULATED\_JOHNSON\_COOK

### **Development history of MAT\_224**

- R4-52798 (june 2009) had the first official release of MAT\_224, load curve driven failure only
- R4-54523: implementation of the NUMINP option by Tobias Erhardt from Dynamore
- R4-54921: implementation of tabulated failure by Paul Du Bois and Murat Buyuk
- R5-59419: (july 2010) QA at LSTC by Gunther Blankenhorn
- Further development driven by demand from industrial users :
- R5-63444: (april 2011) optional coupling with EOS
- R6-68028 : E-modulus can be a function of temperature
- R6-68055 : (july 2011) MAT\_224 is coupled with type 13 tetrahedrons
- R6-70523: ( december 2011 ) tabulated regularisation
- R6-71932 : coupled with \*CONSTRAINED\_TIED\_NODES\_FAILURE
- R6-72543: (february 2012) NUMINT=-200 means no erosion will occur if d=1
- Dev-87525 : (february 2014) BETA<0 refers to a load curve in function of strain rate</p>

# Development of a material and failure model for Ti-6-4

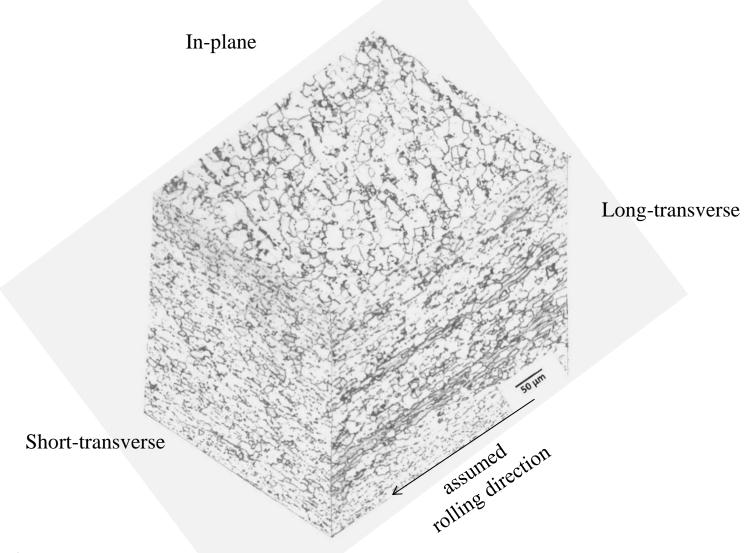


FAA-OSU-GWU-NASA Columbus 2007

Part 1: material model

#### **Careful material selection**

- Material properties depend upon the microstructure : grain size, texture, orientation, surface quality...
- Manufacturing operations (rolling, annealing) influence the microstructure and therefore the material properties
- Specific material models may be needed for individual components
- Some microstructural aspects of the material can be studied under the electronmicroscope



BLM 45 0.530" Ti-6-4

#### Ti-6-4 Texture

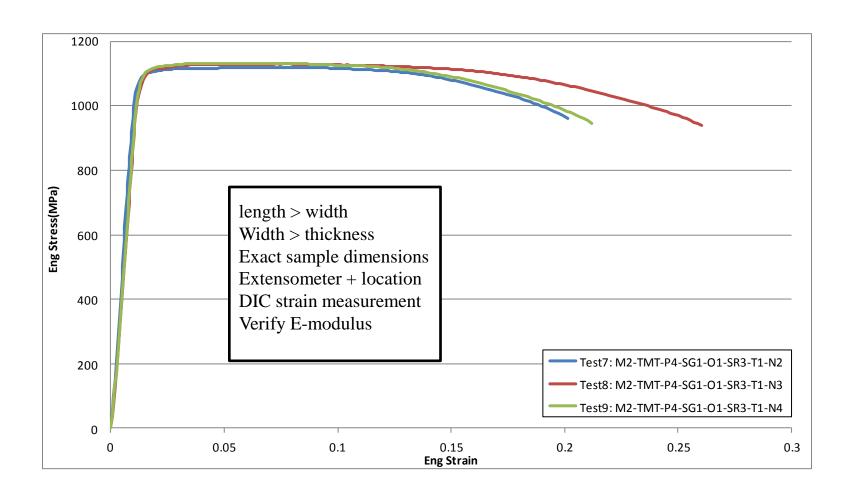
Sample	Texture	Predominant	Degrees off-center
	Index	Texture Type	
BLM 45 0.270"	1.24	Transverse	31
BLM 45 0.530"	1.13	Basal	27
BLM 46 0.270"	1.33	Transverse	24
BLM 47 0.425"	1.54	Transverse	-
0.09"	1.40	Transverse	-
0.135"	1.17	Basal	30
0.25"	1.48	Basal	18
0.50"	1.03	Basal, Transverse	-

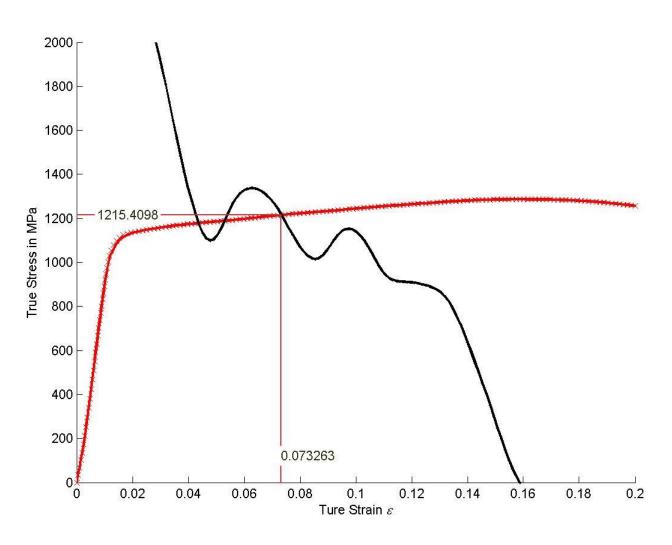
• **Texture index** denotes degree of anisotropy

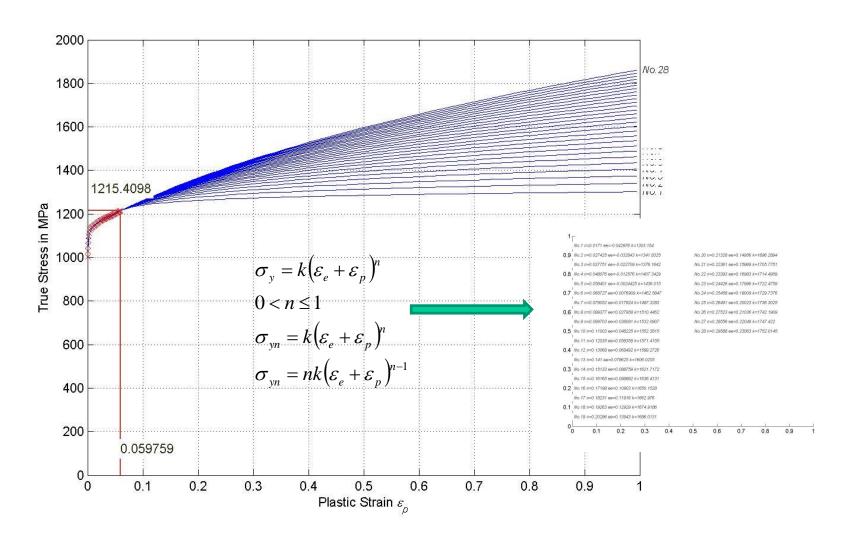
Index ranges from 1 (isotropic) to  $\infty$  (single crystal)

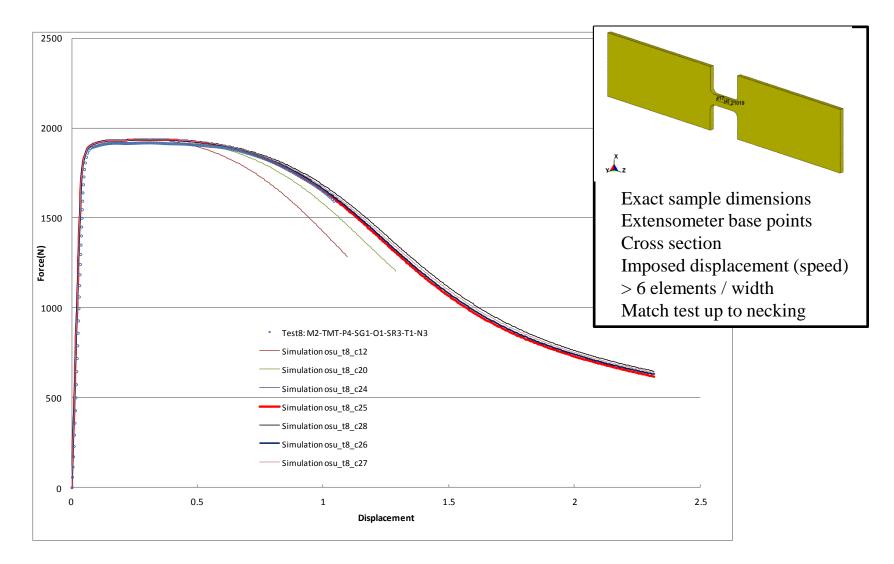
Aluminum foil (highly textured) has an index of 4.14

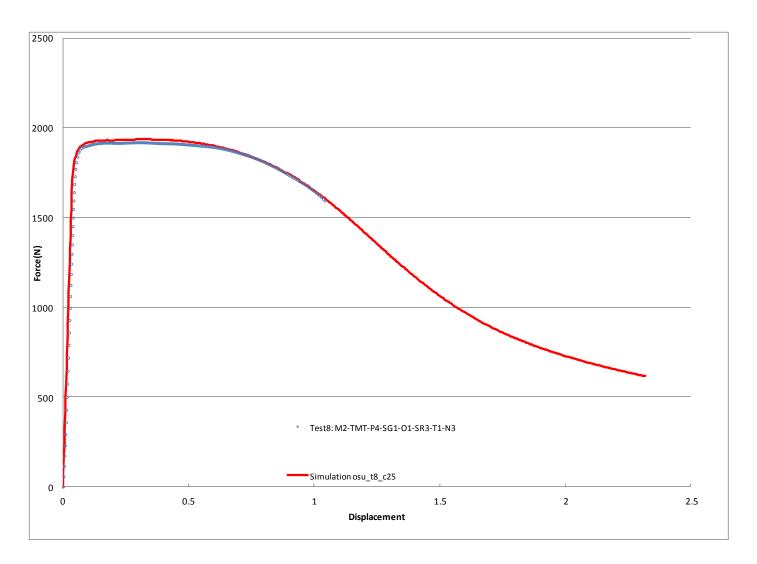
## Example of Ti-6-4 0.5" plate stock at strain rate = 1/s

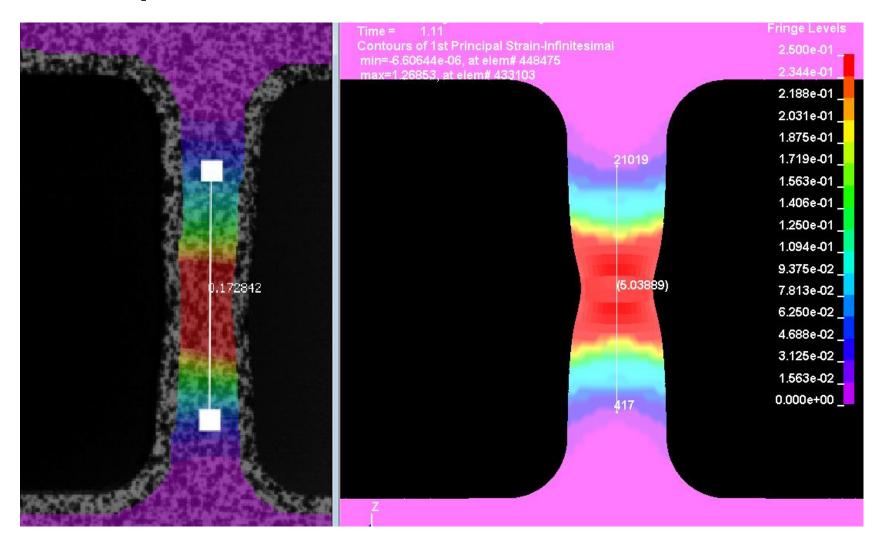


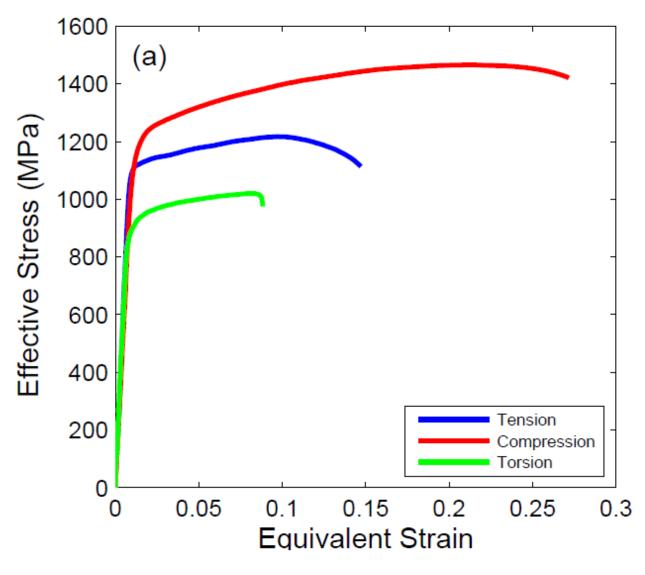






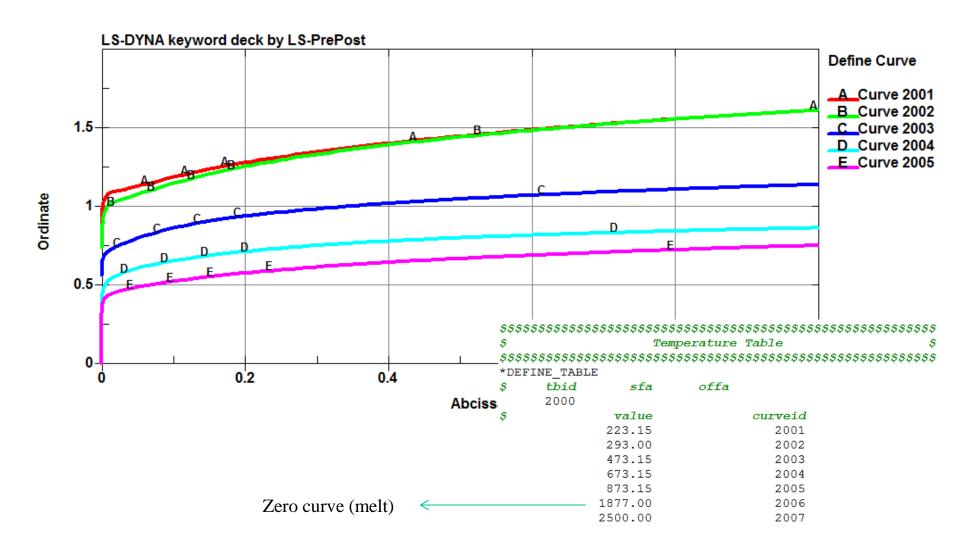




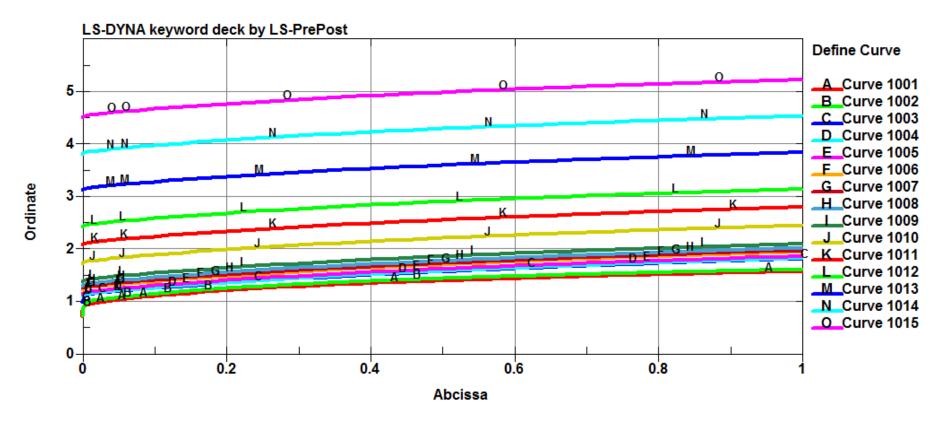


- Plate asymmetry
- MAT\_224 uses tension data to generate yield curve
  - Tension/Torsion will be over estimated
  - Compression/Tor sion will be under estimated
- MAT\_224\_GYS allows for asymmetrical input

# **Temperature Curves (table 2000)**



# **Strain Rate Curves (table 1000)**

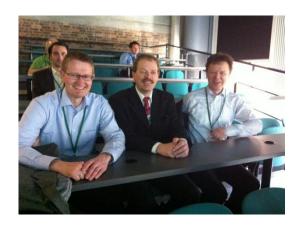


**Numerical Simulation of Dynamic Failure: ballistic tests on Titanium-6Al-4V** 

### conclusion

- Realistic material data are needed beyond necking (localisation) and up to failure
  if a predictive simulation with respect to failure is to be achieved
- A procedure was established allowing for reasonable accuracy as well as cost efficiency
- Value of DIC cannot be overestimated

# adiabatic instability



**Shear bands Adiabatic vs isothermal instabilities** 

- shear bands seem to be the main instability that is observed in high velocity impact phenomena
- plugging is caused by a thermoplastic shear instability, (so is the fragmentation of an exploding cylinder)

#### Adiabatic instabilities: Shear bands

- Typical shear band widths are 10 to 100 microns
- High local values of shear strain: 5 to 100
- Ultra high local shear strain rates: 10000/s to 1000000/s
- Local temperature rises several hundred degrees
- High propagation speeds : typically 1000 m/s
- Not a crack
- Shear bands are precursors to rupture
- Cfr. Tzavaras

#### Adiabatic instabilities: Localisation mechanism

- Under isothermal conditions metals strain harden
- Under high strain rates conditions change from isothermal to adiabatic
- If heat diffusion is too slow to equalize temperatures in the timescale of the loading then shear bands may occur
- Non-uniform strains induce non-uniform heating
- Hot spots are softer, cold spots are harder, this amplifies the non-uniformity of the strain field
- Two effects oppose the instability: heat diffusion and momentum diffusion induced by strain rate dependency

 Assume an elasto-plastic material with strain hardening and linear thermal softening:

$$\sigma_y = f(\varepsilon_p)h(T) = f(\varepsilon_p)\left(1 - \frac{T - T_R}{T_M - T_R}\right)$$

• The maximum force criterion under plane stress conditions gives :

$$\frac{d\sigma_{y}}{d\varepsilon_{p}} = \sigma_{y} \frac{\left(4 - 3a - 3a^{2} + 4a^{3}\right)}{4\left(1 + a^{2} - a\right)^{3/2}} \qquad a = \frac{\sigma_{2}}{\sigma_{1}}$$

• With special cases for uniaxial tension and pure shear :

$$\frac{d\sigma_{y}}{d\varepsilon_{p}} = \sigma_{y} \qquad a = 0$$

$$\frac{d\sigma_{y}}{d\varepsilon_{p}} = 0 \qquad a = -1$$

 The previous formulas contain total derivatives: yield stress is considered a function of equivalent plastic strain AND temperature

We can derive an adiabatic instability condition :

$$\frac{d\sigma_{y}}{d\varepsilon_{p}} = \sigma_{y} \frac{\left(4 - 3a - 3a^{2} + 4a^{3}\right)}{4\left(1 + a^{2} - a\right)^{3/2}} \\
\frac{d\sigma_{y}}{d\varepsilon_{p}} = \frac{\partial\sigma_{y}}{\partial\varepsilon_{p}} + \frac{\partial\sigma_{y}}{\partial T} \frac{dT}{d\varepsilon_{p}}$$

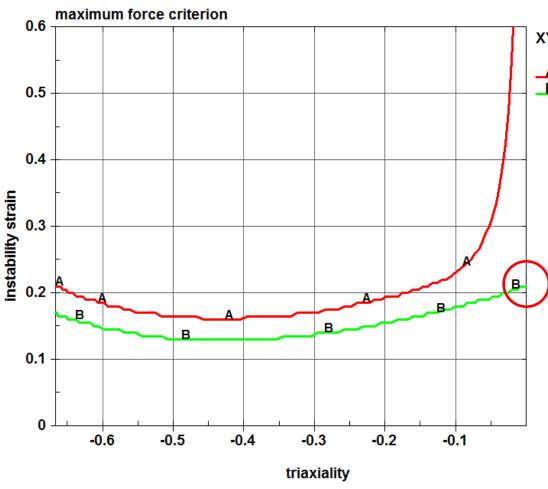
$$\Rightarrow \frac{\partial\sigma_{y}}{\partial\varepsilon_{p}} = \sigma_{y} \frac{\left(4 - 3a - 3a^{2} + 4a^{3}\right)}{4\left(1 + a^{2} - a\right)^{3/2}} - \frac{\partial\sigma_{y}}{\partial T} \frac{dT}{d\varepsilon_{p}}$$

$$dT = \frac{dW}{\rho C_{p}} = \frac{\sigma_{y} d\varepsilon_{p}}{\rho C_{p}} \Rightarrow \frac{dT}{d\varepsilon_{p}} = \frac{\sigma_{y}}{\rho C_{p}}$$

$$\frac{\partial \sigma_{y}}{\partial T} = -\frac{f(\varepsilon_{p})}{T_{M} - T_{R}}$$

$$\frac{\partial \sigma_{y}}{\partial \varepsilon_{p}} = \frac{\partial f(\varepsilon_{p})}{\partial \varepsilon_{p}} \frac{T - T_{R}}{T_{M} - T_{R}}$$

$$\Rightarrow \frac{\partial f(\varepsilon_{p})}{\partial \varepsilon_{p}} = f(\varepsilon_{p}) \frac{(4 - 3a - 3a^{2} + 4a^{3})}{4(1 + a^{2} - a)^{3/2}} + \frac{f^{2}(\varepsilon_{p})}{(T_{M} - T_{R})\rho C_{p}}$$



XY data

A\_isothermal B\_adiabatic

Onset of instability for a strain hardening elasto-plastic material (Al2024) with

linear temperature dependency:

$$\sigma_{y} = f(\varepsilon_{p}) \frac{T - T_{R}}{T_{M} - T_{R}}$$

(assuming small elastic strains and plane stress )

 Analytical solution for the onset of shear banding in the case of a Johnosn-Cook law with linear temperature dependency :

$$\frac{d\sigma_{y}}{d\varepsilon_{p}} = 0$$

$$\frac{d\sigma_{y}}{d\varepsilon_{p}} = \frac{\partial\sigma_{y}}{\partial\varepsilon_{p}} + \frac{\partial\sigma_{y}}{\partial T} \frac{dT}{d\varepsilon_{p}}$$

$$\Rightarrow \frac{\partial\sigma_{y}}{\partial\varepsilon_{p}} = -\frac{\partial\sigma_{y}}{\partial T} \frac{dT}{d\varepsilon_{p}}$$

$$dT = \frac{dW}{\rho C_p} = \frac{\sigma_y d\varepsilon_p}{\rho C_p} \Rightarrow \frac{dT}{d\varepsilon_p} = \frac{a + b\varepsilon_p^n}{\rho C_p}$$
$$\frac{\partial \sigma_y}{\partial T} = -\frac{a + b\varepsilon_p^n}{T_M - T_R} \left(\varepsilon_p = cte\right)$$

$$\Rightarrow \frac{\partial \sigma_{y}}{\partial \varepsilon_{p}} = -\frac{\partial \sigma_{y}}{\partial T} \frac{dT}{d\varepsilon_{p}} \Rightarrow bn \varepsilon_{p}^{n-1} = \frac{\left(a + b \varepsilon_{p}^{n}\right)^{2}}{\left(T_{M} - T_{R}\right) \rho C}$$

 Analytical solution for the onset of adiabatic diffuse necking in the case of a Johnosn-Cook law with linear temperature dependency:

$$\frac{d\sigma_{y}}{d\varepsilon_{p}} = \sigma_{y}$$

$$\frac{d\sigma_{y}}{d\varepsilon_{p}} = \frac{\partial\sigma_{y}}{\partial\varepsilon_{p}} + \frac{\partial\sigma_{y}}{\partial T} \frac{dT}{d\varepsilon_{p}}$$

$$\Rightarrow \frac{\partial\sigma_{y}}{\partial\varepsilon_{p}} = \sigma_{y} - \frac{\partial\sigma_{y}}{\partial T} \frac{dT}{d\varepsilon_{p}}$$

$$dT = \frac{dW}{\rho C_p} = \frac{\sigma_y d\varepsilon_p}{\rho C_p} \Rightarrow \frac{dT}{d\varepsilon_p} = \frac{a + b\varepsilon_p^n}{\rho C_p}$$
$$\frac{\partial \sigma_y}{\partial T} = -\frac{a + b\varepsilon_p^n}{T_M - T_R} \left(\varepsilon_p = cte\right)$$

$$\Rightarrow \frac{\partial \sigma_{y}}{\partial \varepsilon_{p}} = \sigma_{y} - \frac{\partial \sigma_{y}}{\partial T} \frac{dT}{d\varepsilon_{p}} \Rightarrow bn\varepsilon_{p}^{n-1} = \left(a + b\varepsilon_{p}^{n}\right) + \frac{\left(a + b\varepsilon_{p}^{n}\right)^{2}}{\left(T_{M} - T_{R}\right)\rho C}$$

#### Material instabilities and strain localisation

example for quasistatic localisation plastic strain values :

$$\begin{array}{c} a = 0.2GPa \\ b = 0.6GPa \\ n = 0.4 \\ T_{M} - T_{R} = 775 - 293 \\ \rho = 7.8E - 6kg / mm^{3} \\ C_{p} = 445kNmm / kg^{\circ}K \end{array} \} \Rightarrow \begin{cases} \varepsilon_{p} = 0.254 & \text{isothermal tension} \\ \varepsilon_{p} = 0.186 & \text{adiabatic tension} \\ \varepsilon_{p} = 0.299 & \text{adiabatic shear} \end{cases}$$

### conclusion

- Triaxiality dependent regularisation needs to be adapted to the conditions of the impact (temperature, rate...)
- Every material needs an individual assessment with respect to occurrence of plastic instability
- For a predictive simulation with respect to failure, the mesh size must take the width of the shear band into account
- The width of the shear band depends a.o. on the hardness and strength of the material ( see reference )
- Depending on the width of the shear band ( 10 to 50 micron ) the number of elements over the thickness of the target plate needed for a failure simulation can be 100 or more ( see reference )

**Numerical Simulation of Dynamic Failure: ballistic tests on Titanium-6Al-4V** 

### references

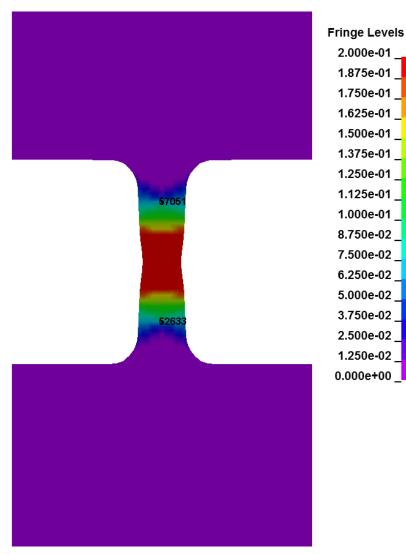
 Are numerical simulations of ballistic impact predictive? Borvik, Hopperstad, Langseth, Berstad, 2011

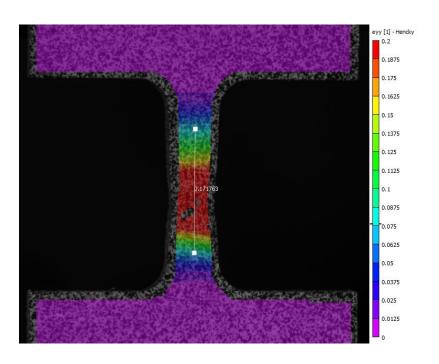
# Development of a material and failure model for Ti-6-4

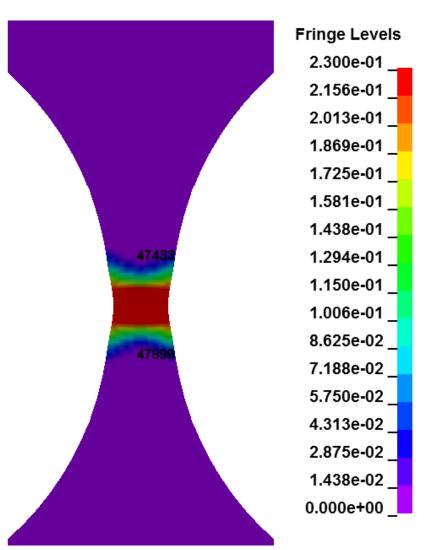


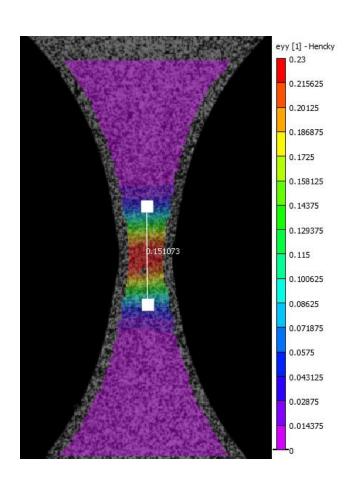
FAA-OSU-GWU-NASA Columbus 2007

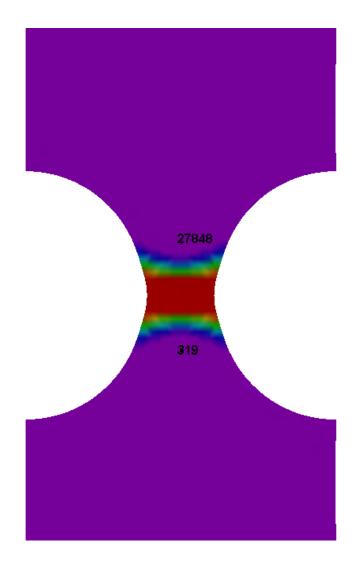
Part 2 : failure model

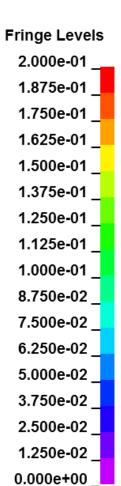


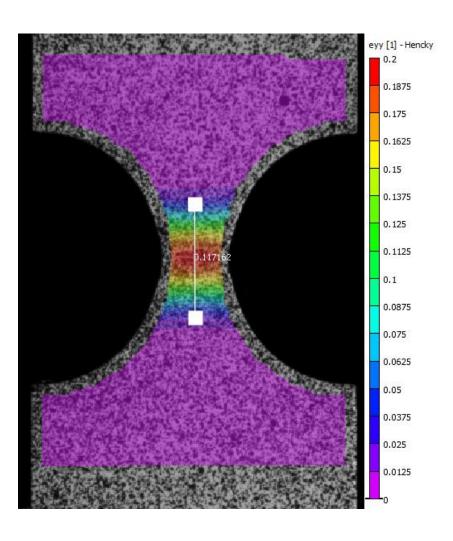


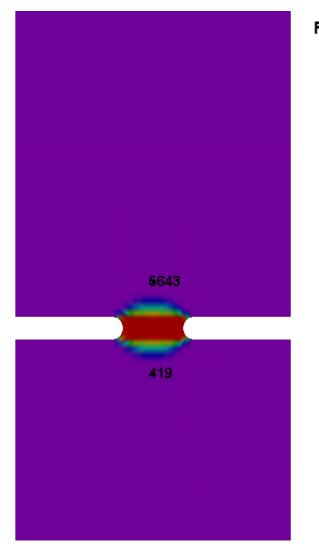


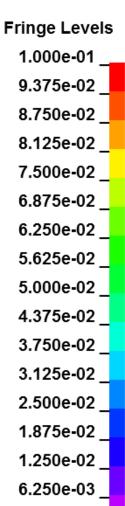




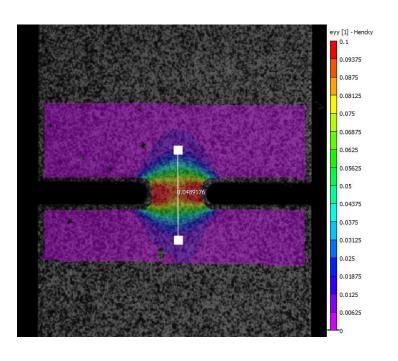


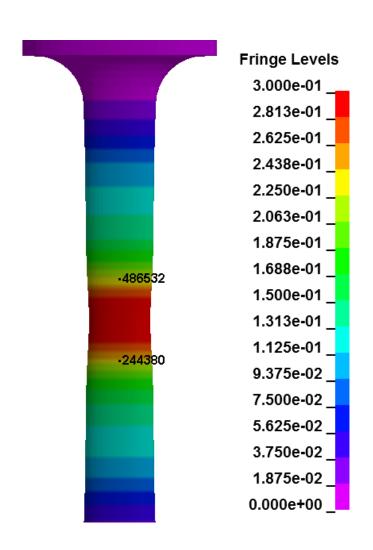


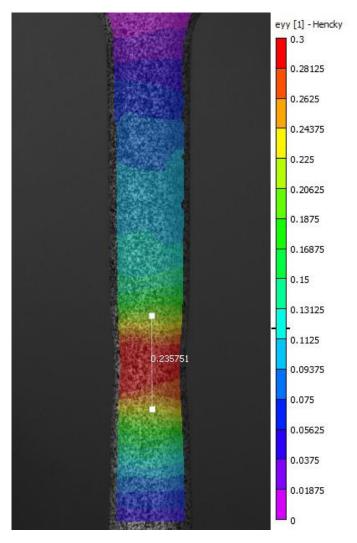


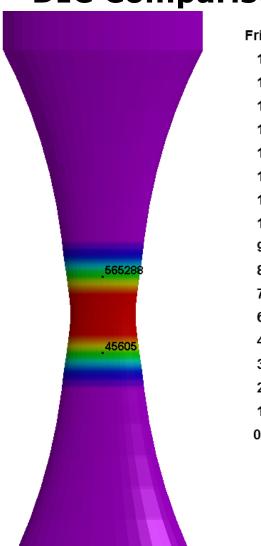


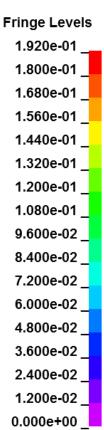
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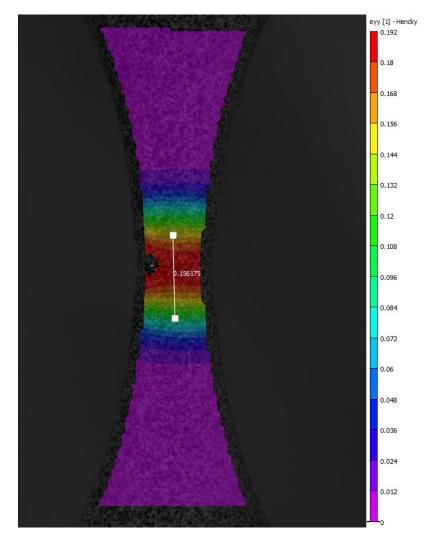


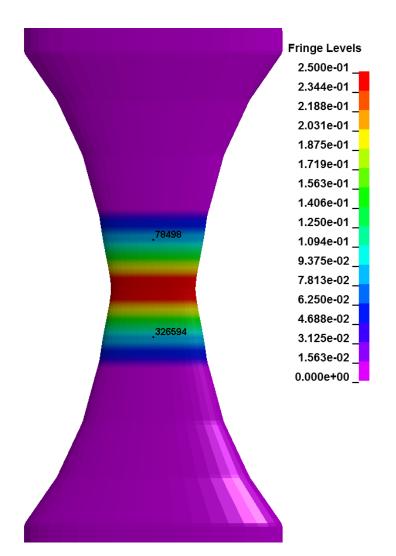


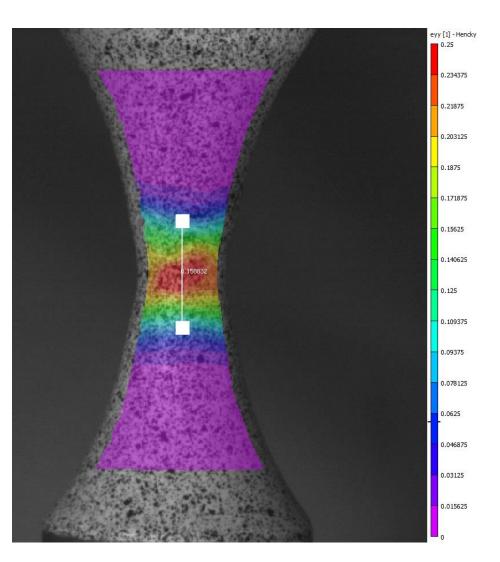


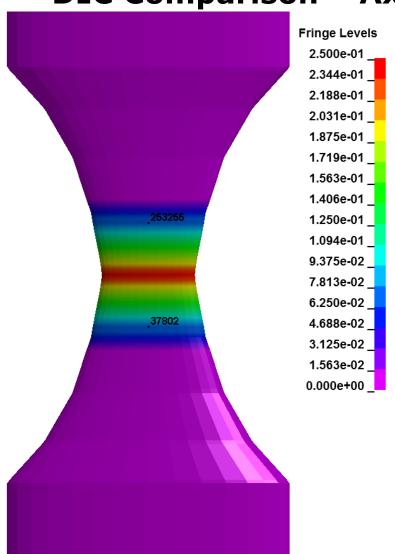


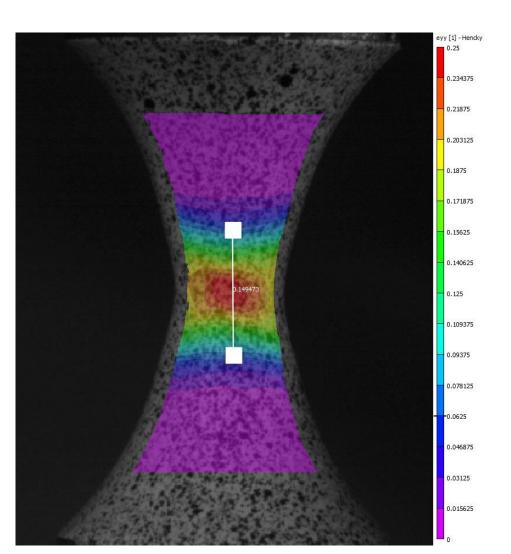


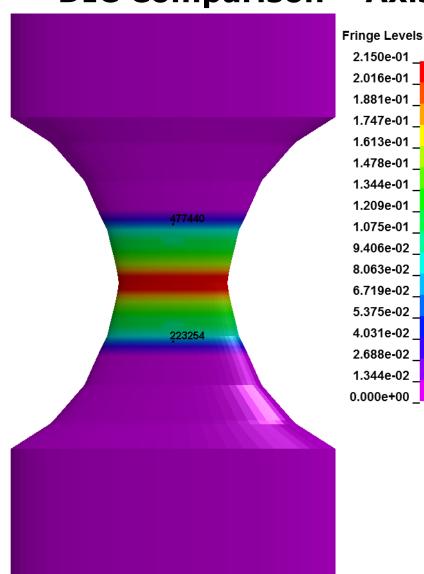


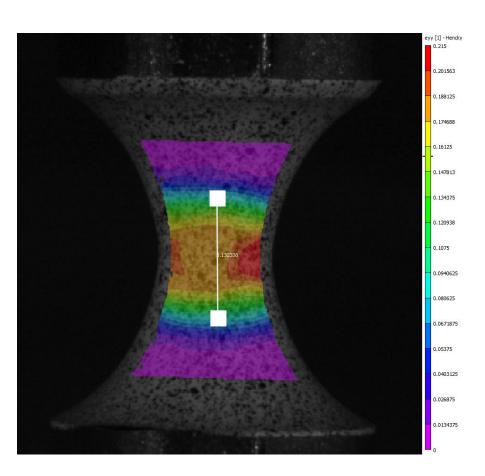


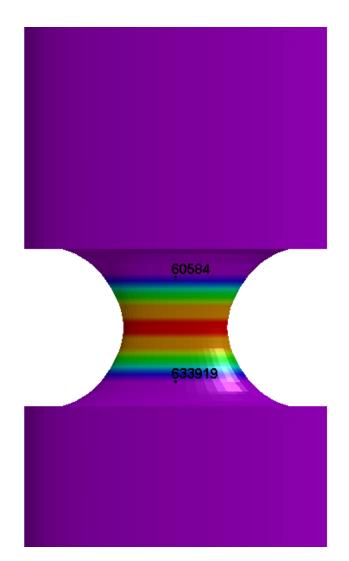












#### Fringe Levels

1.800e-01

1.688e-01

1.575e-01 \_

1.463e-01

1.350e-01 \_

1.238e-01

1.125e-01 \_

1.013e-01

9.000e-02

7.875e-02

6.750e-02 \_

5.625e-02

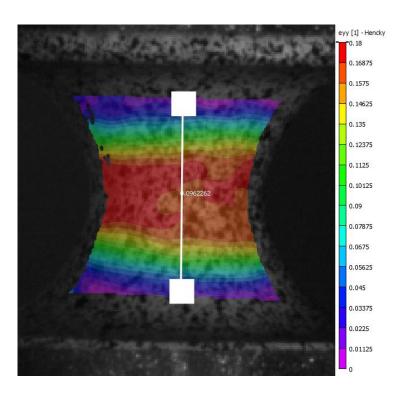
4.500e-02

3.375e-02

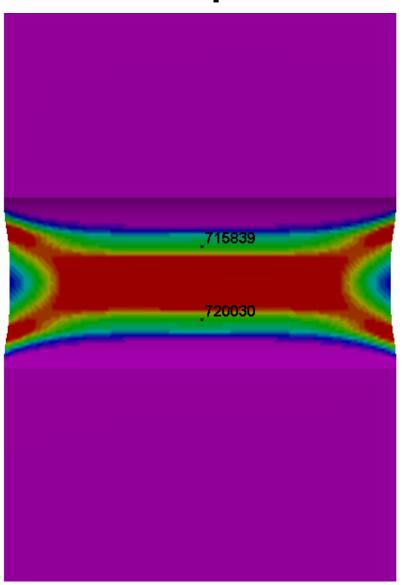
2.250e-02

1.125e-02

0.000e+00 \_



# **DIC Comparison – Plane Strain 1**



#### Fringe Levels

1.500e-01

1.406e-01

1.313e-01

1.219e-01

1.125e-01

1.031e-01

9.375e-02

8.438e-02

7.500e-02

6.563e-02

5.625e-02

4.688e-02

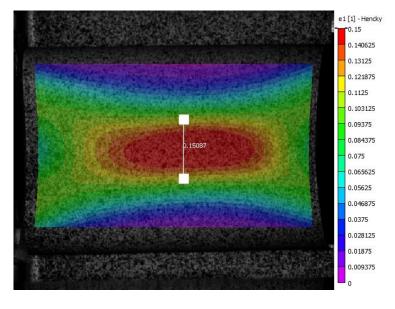
3.750e-02

2.813e-02

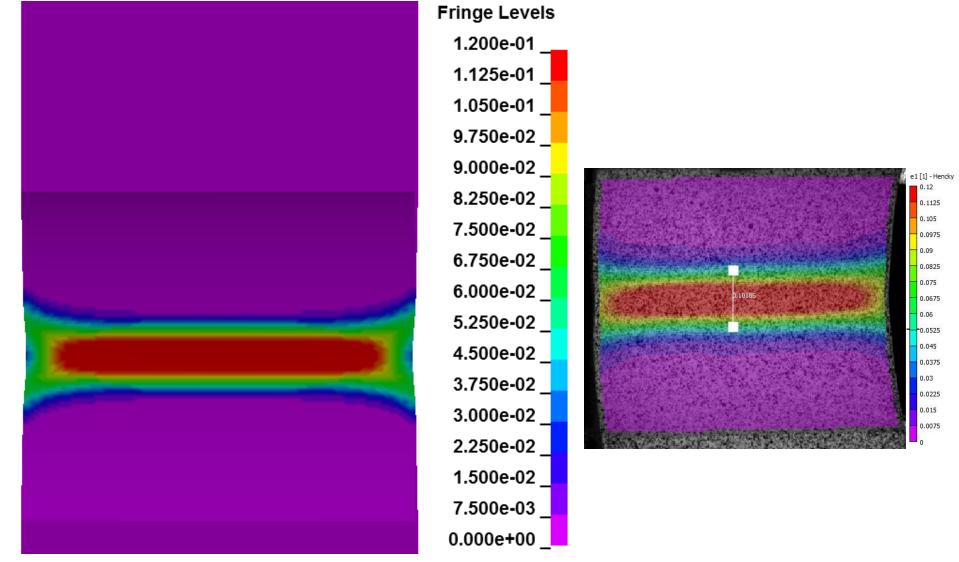
1.875e-02

9.375e-03

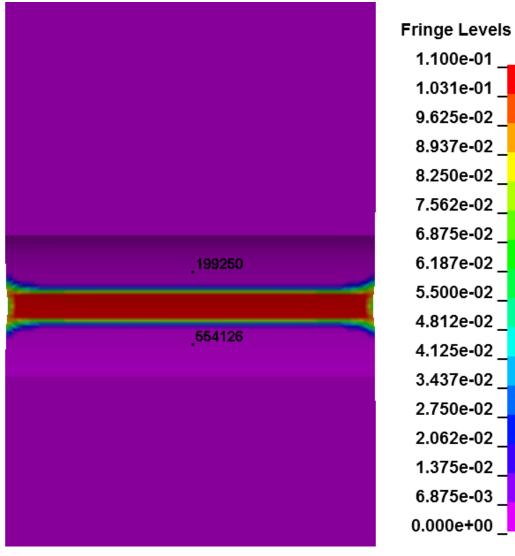
0.000e+00

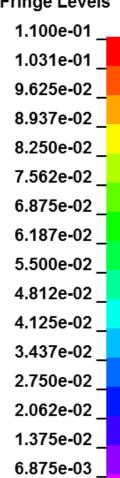


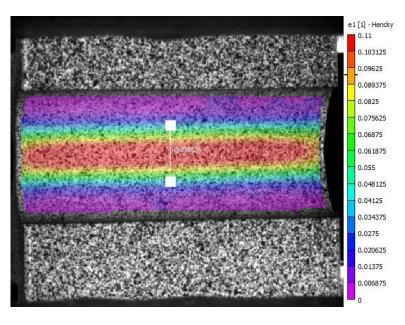
# **DIC Comparison – Plane Strain 2**

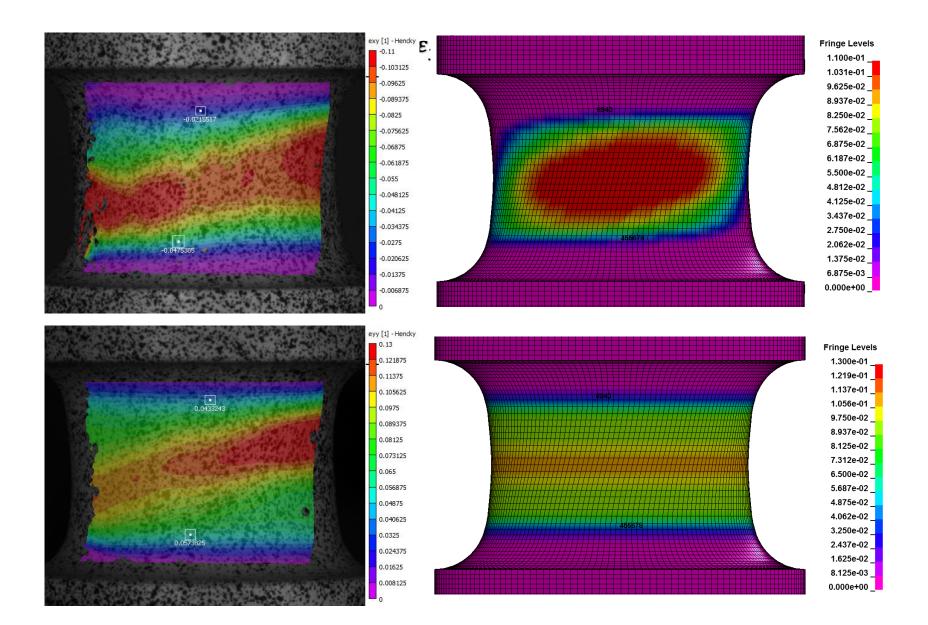


# **DIC Comparison – Plane Strain 3**

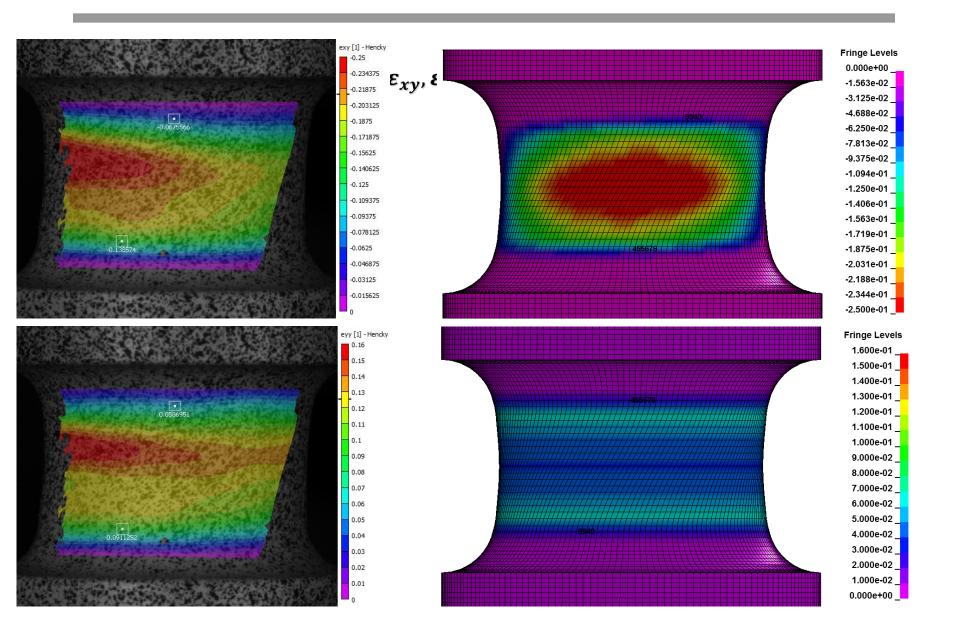


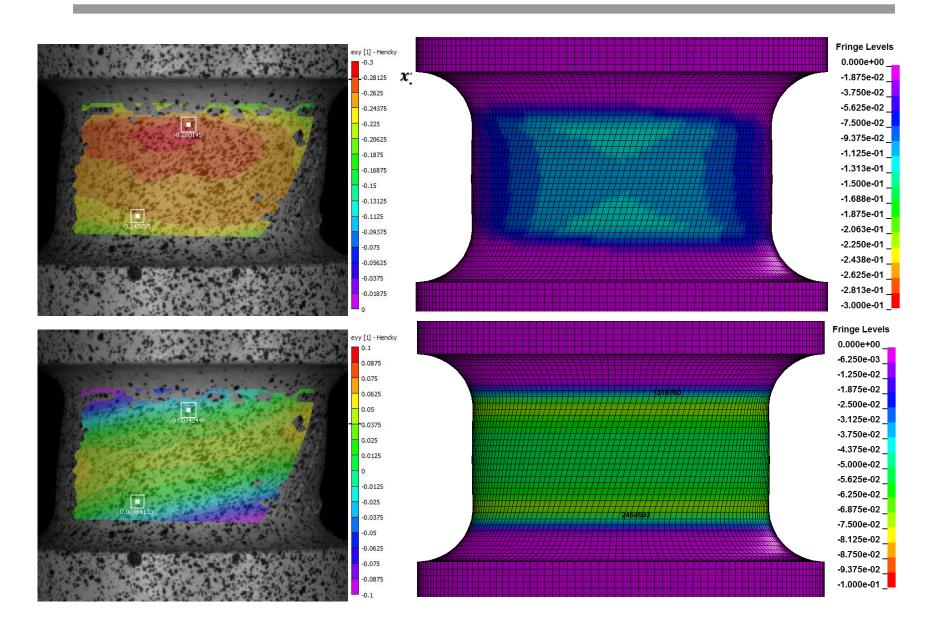






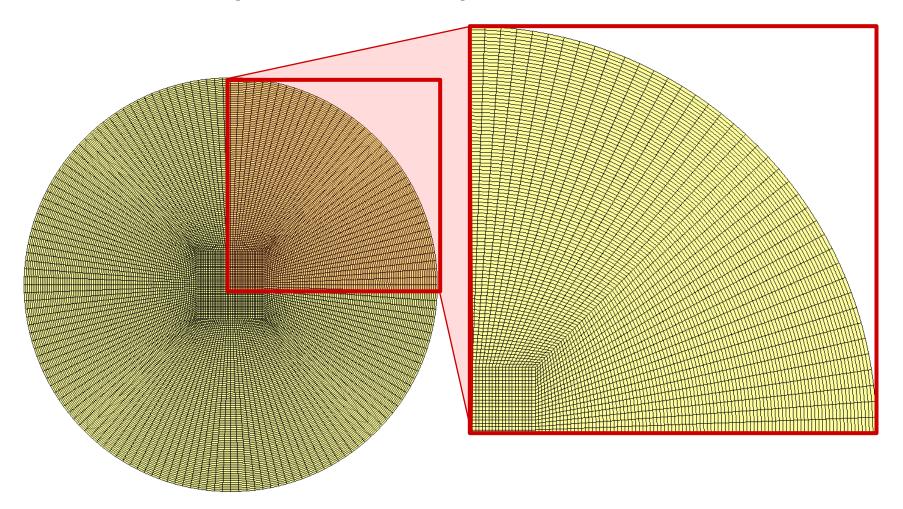
#### **Numerical Simulation of Dynamic Failure: ballistic tests on Titanium-6Al-4V**

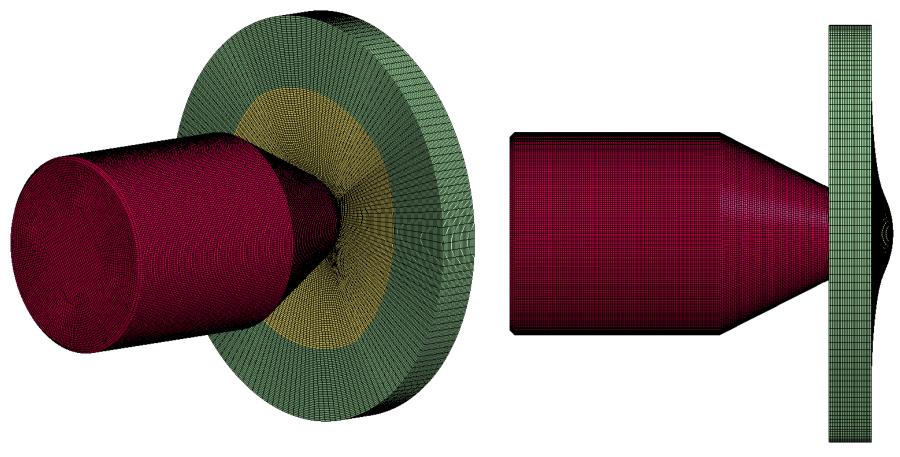


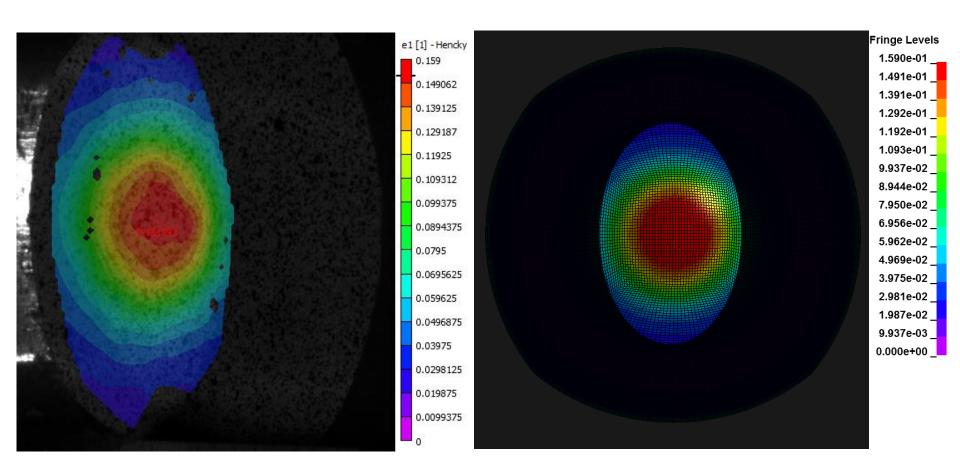


# Large diameter punch tests: Specimen Geometry

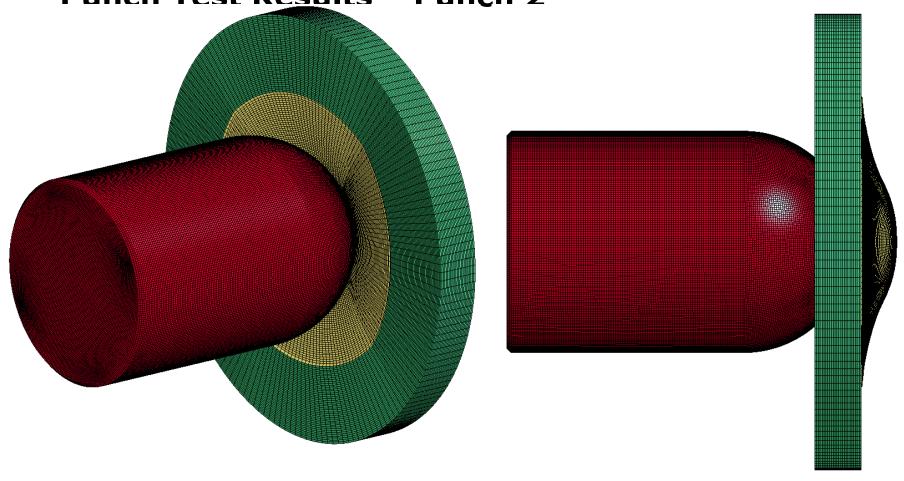
420,000 elements (0.2 mm element size)

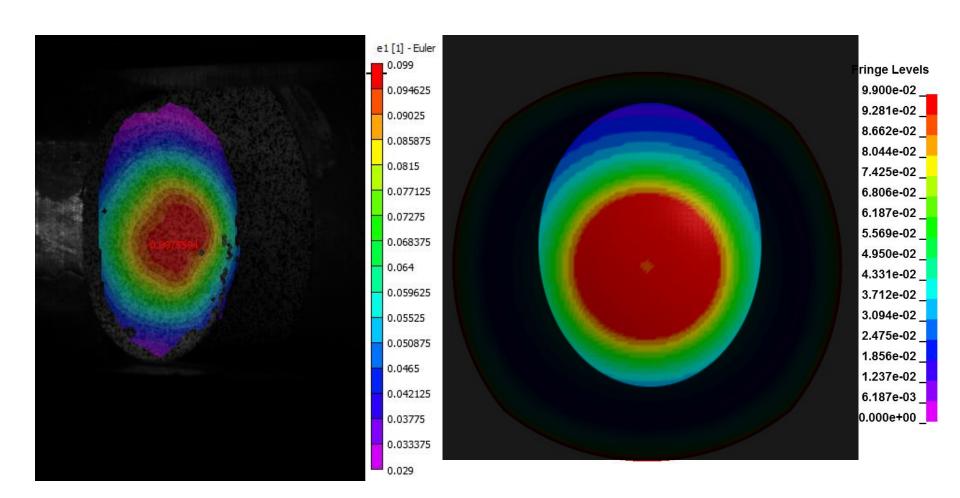


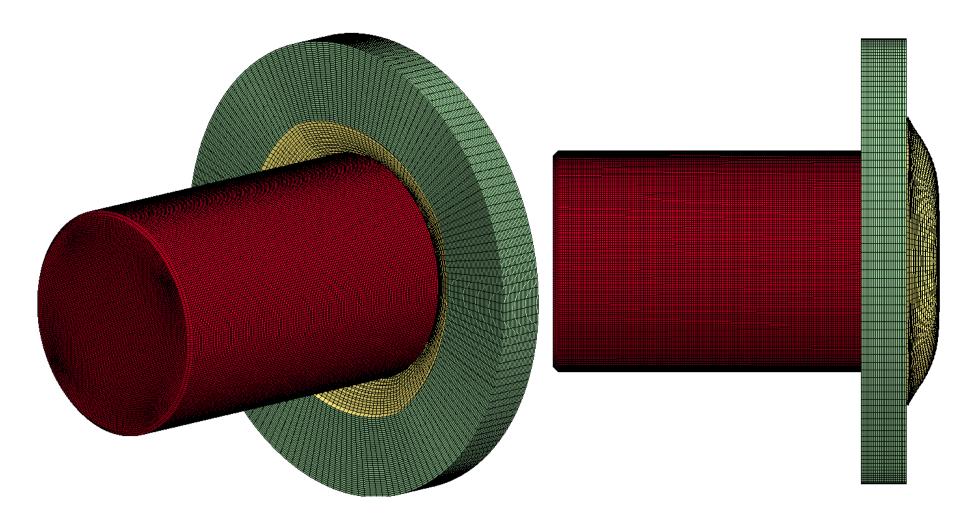


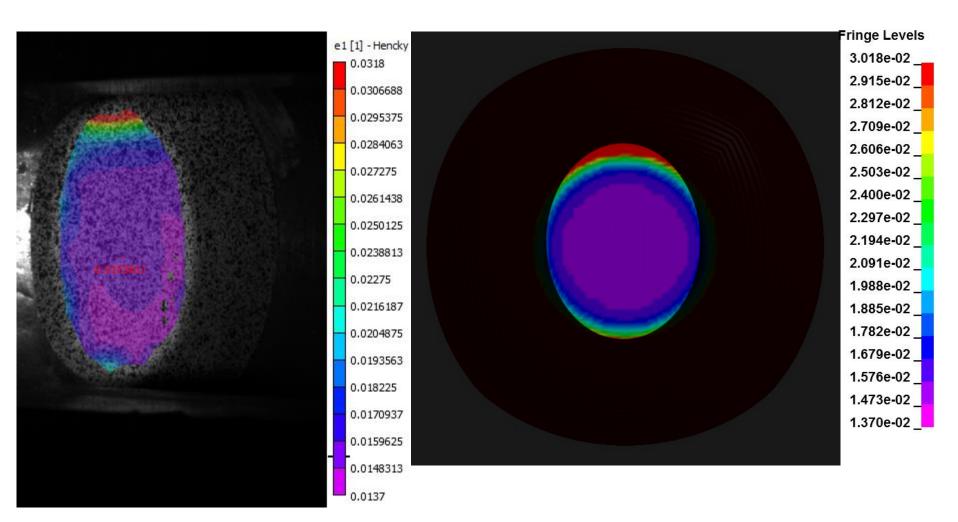










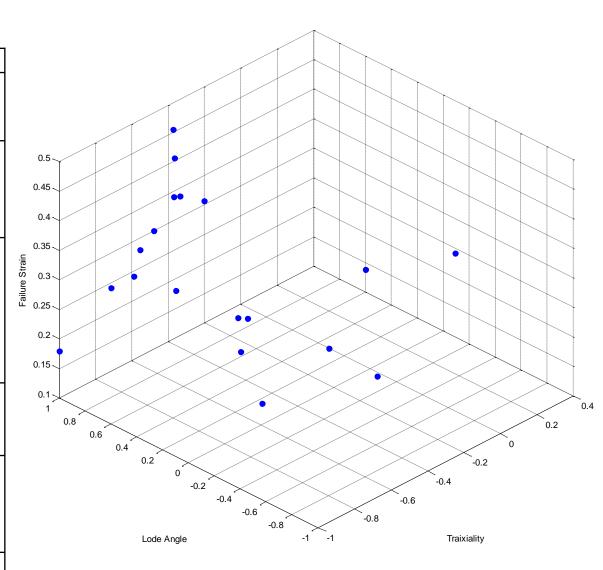


# **Completed Failure Table**

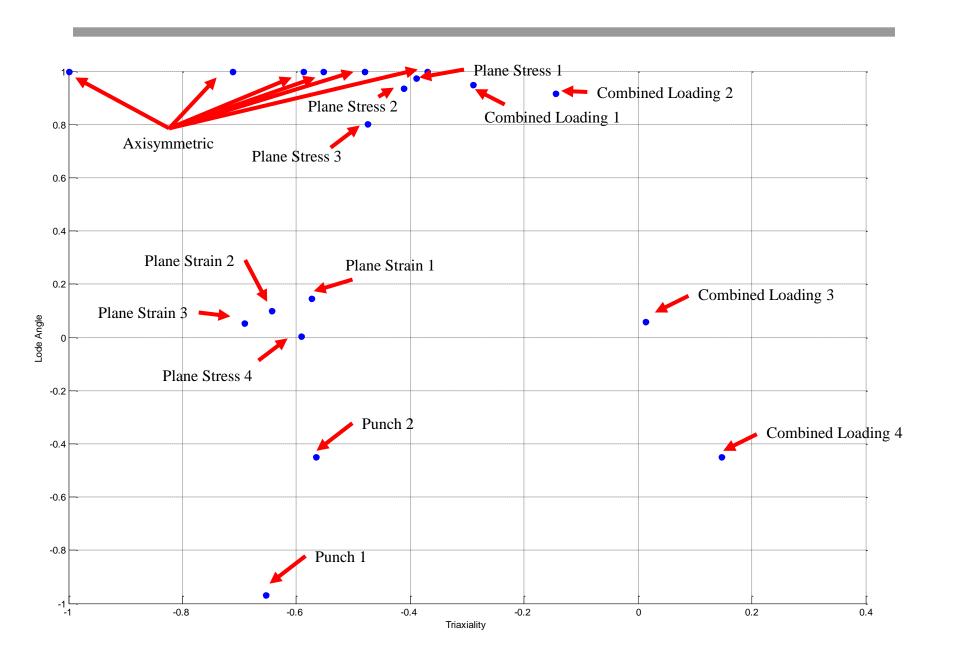
Test Number	Geometry	OSU			GMU		
		Triaxiality	Lode	Failure	Triaxialit	Lode	Failure
			Parameter	Strain	У	Parameter	Strain
SG1	Plane Stress	0.400	0.850	0.590	0.390	0.975	0.460
SG2		0.431	0.719	0.440	0.412	0.935	0.420
SG3		0.489	0.528	0.430	0.475	0.803	0.380
SG4		0.583	0.014	0.140	0.592	0.005	0.135
SG5	Axisymmetric	0.369	1.000	0.310	0.370	1.000	0.340
SG6		0.492	1.000	0.310	0.480	1.000	0.300
SG7		0.564	1.000	0.320	0.553	1.000	0.280
SG8		0.618	1.000	0.270	0.588	1.000	0.240
SG9		0.751	1.000	0.270	0.712	1.000	0.240
SG10		0.956	1.000	0.220	1.000	1.000	0.180
SG11	Plane Strain	0.470	0.506	0.210	0.573	0.146	0.260
SG12		0.660	0.040	0.220	0.643	0.099	0.220
SG13		0.768	0.025	0.210	0.691	0.054	0.290
LR1	Combined Loading	0.252	0.706	0.290	0.289	0.949	0.191
LR2		0.150	0.400	0.510	0.145	0.917	0.325
LR3		0.000	0.000	0.430	0.014	0.059	0.259
LR4		-0.148	-0.394	0.420	-0.147	-0.450	0.321
Punch1					0.653	-0.969	0.297
Punch2	Punch Tests				0.565	-0.475	0.276
Punch3					-0.077	-0.073	0.766

# **Data points**

	GMU					
Test Number	Triaxialit y	Lode Paramet er	Failure Strain			
SG1	0.390	0.975	0.460			
SG2	0.412	0.935	0.420			
SG3	0.475	0.803	0.380			
SG4	0.592	0.005	0.135			
SG5	0.370	1.000	0.340			
SG6	0.480	1.000	0.300			
SG7	0.553	1.000	0.280			
SG8	0.588	1.000	0.240			
SG9	0.712	1.000	0.240			
SG10	1.000	1.000	0.180			
SG11	0.573	0.146	0.260			
SG12	0.643	0.099	0.220			
SG13	0.691	0.054	0.290			
LR1	0.289	0.949	0.191			
LR2	0.145	0.917	0.325			
LR3	0.014	0.059	0.259			
LR4	-0.147	-0.450	0.321			
Punch1	0.653	-0.969	0.297			
Punch2	0.565	-0.475	0.276			
Punch3	-0.077	-0.073	0.766			



#### **Numerical Simulation of Dynamic Failure : ballistic tests on Titanium-6Al-4V**



### **Failure Surface Plan**

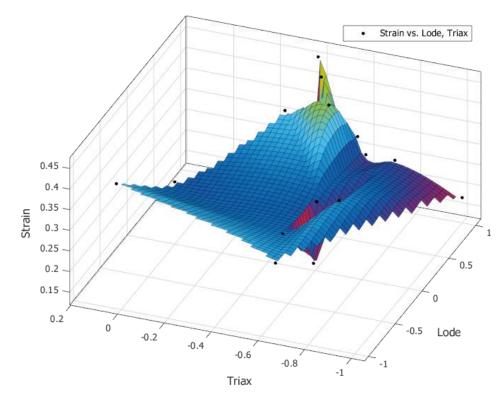
 Automatic surface interpolation methods do not work as well for data sets with large voids and clusters

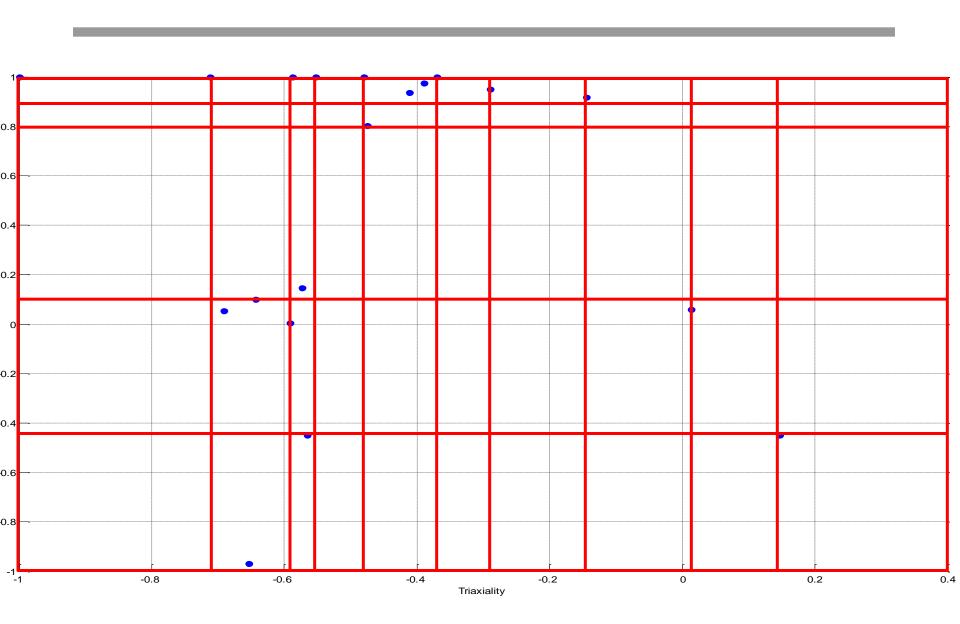
Also very difficult because the boundaries are not defined with discrete

points

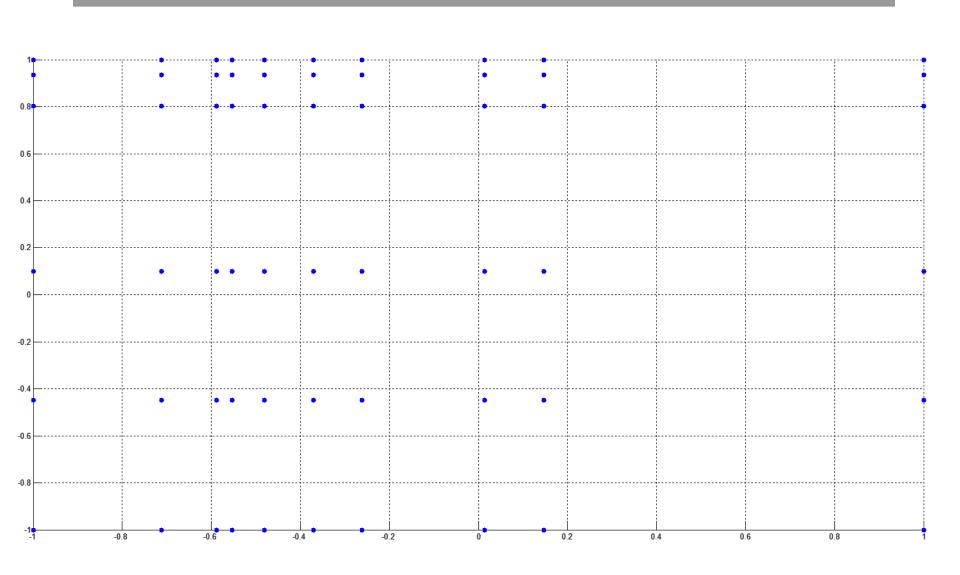
New plan:

- Cluster
- Grid
- •Interpolate





**Cluster and Grid Plan** 

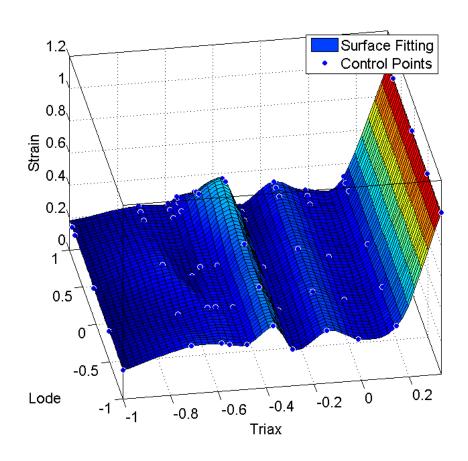


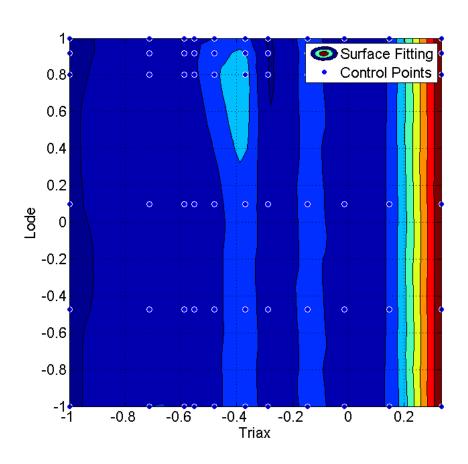
**Grid Points** 

## **Failure Surface Fitting**

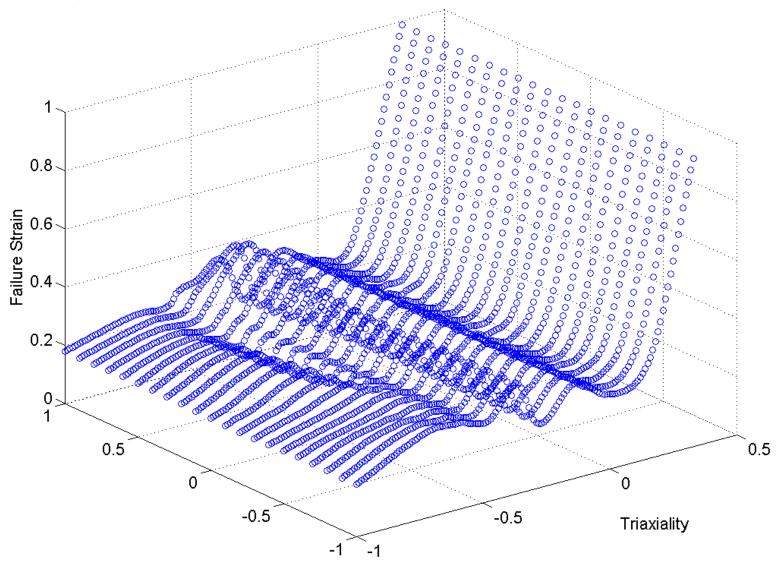
- MATLAB code was written to implement "control point matrix strategy"
- Code implementation:
  - 1. Initializing matrices for Triaxiality, Lode angle, and failure strain
  - 2. Populating Triaxiality and Lode angle matrices with grid points
  - 3. Populating failure strain by selecting nearest test data point
    - This sometimes included averaging two or three nearest test data points
  - 4. Implement Curve Fitting Toolbox generated code for fitting a surface
    - From Triaxiality of -1 to 0.33
    - From Lode angle of -1 to 1
  - 5. Create evaluation points using generated fit and store into matrix
    - Triaxiality: [-1:.01:.33] (every 0.1 units / 134 total points)
    - Lode angle=[-1:.1:1] (every 0.1 units / 21 points)
    - 2814 total points (21 ls-dyna curves)
  - 6. Generate keyword file automatically using evaluation matrices in previous step

### **Surface Fitting Result**

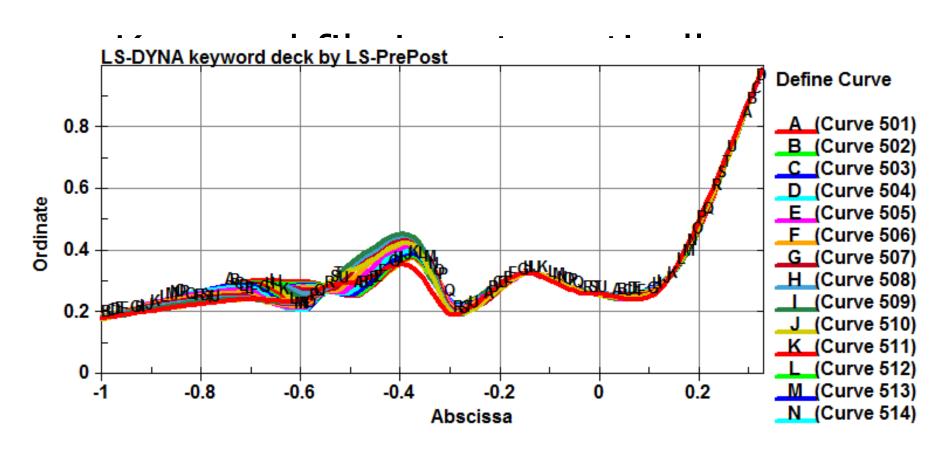




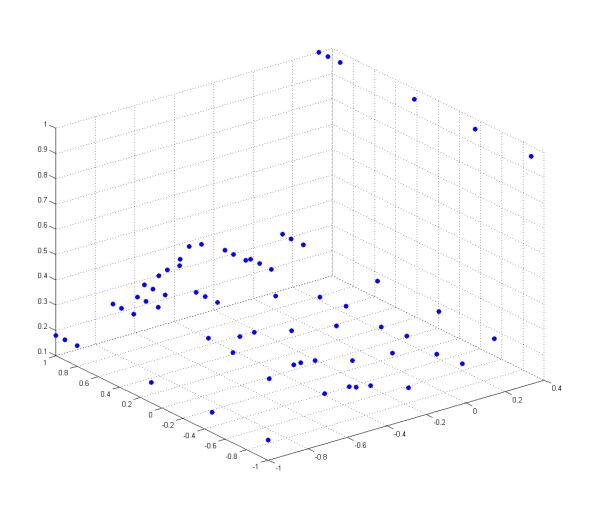
### **Evaluation Points Result**

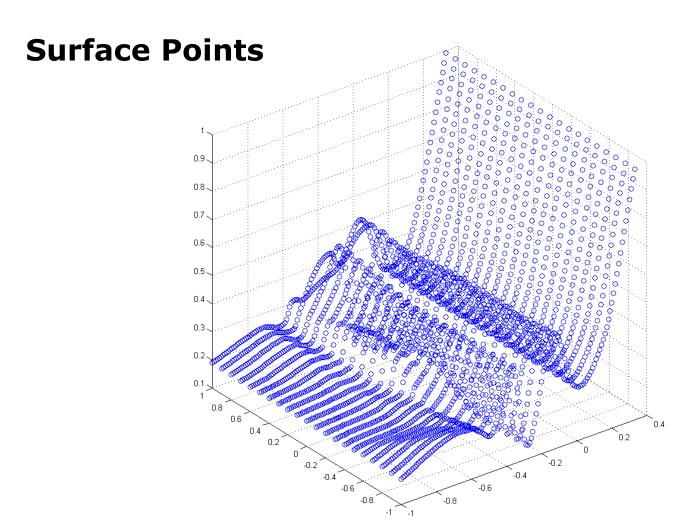


### **Keyword File**

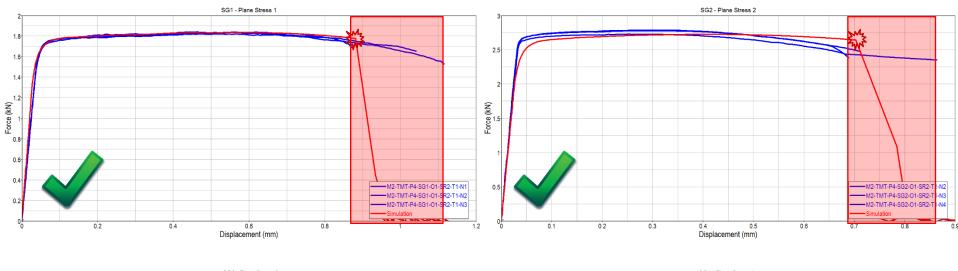


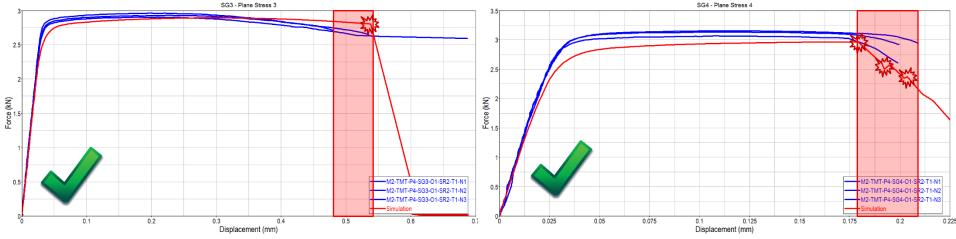
	GMU						
Test #	Triaxiali ty	Lode Paramete r	Failure Strain				
SG1	-0.390	0.975	0.460				
SG2	-0.412	0.935	0.420				
SG3	-0.475	0.803	0.380				
SG4	-0.592	0.005	0.135				
SG5	-0.370	1.000	0.340				
SG6	-0.480	1.000	0.300				
SG7	-0.553	1.000	0.280				
SG8	-0.588	1.000	0.240				
SG9	-0.712	1.000	0.240				
SG10	-1.000	1.000	0.180				
SG11	-0.573	0.146	0.260				
SG12	-0.643	0.099	0.220				
SG13	-0.691	0.054	0.290				
LR1	-0.289	0.949	0.191				
LR2	-0.145	0.917	0.325				
LR3	-0.014	0.059	0.259				
LR4	0.147	-0.450	0.321				
Punch 1	-0.653	-0.969	0.297				
Punch 2	-0.565	-0.475	0.276				





## **Plane Stress**



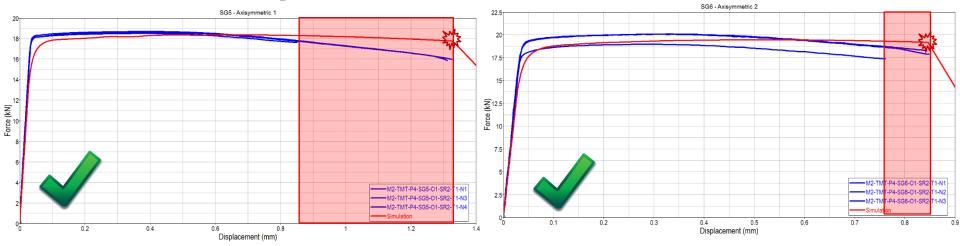


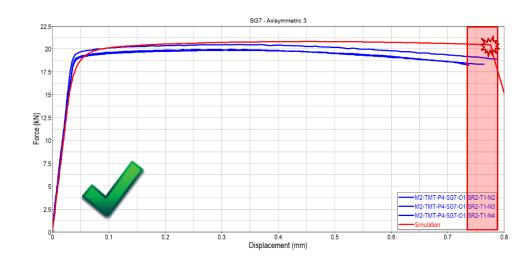


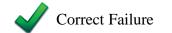


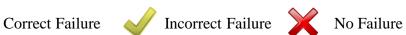


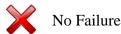
# **Axisymmetric**

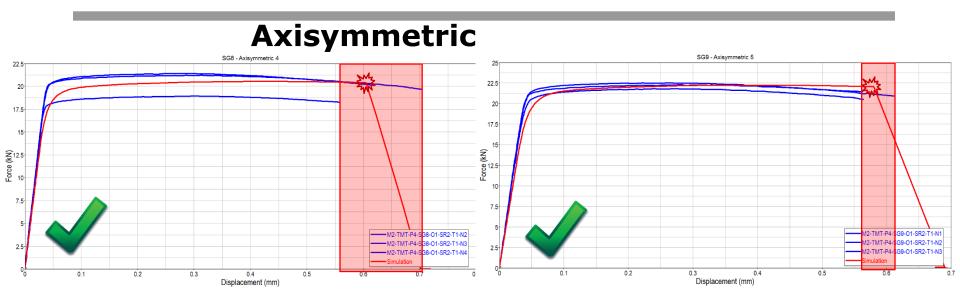


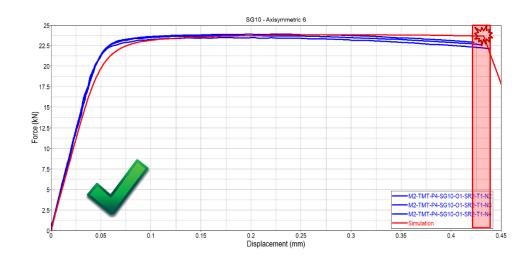


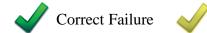




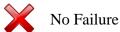




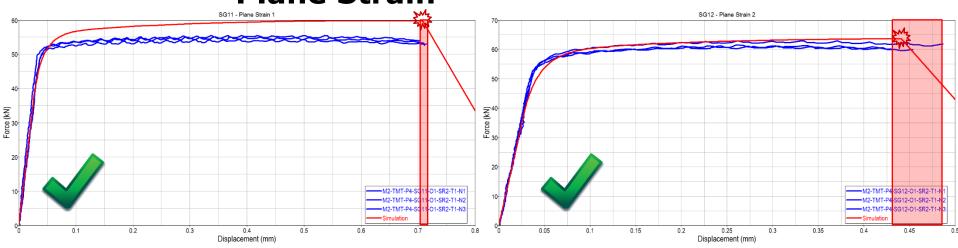


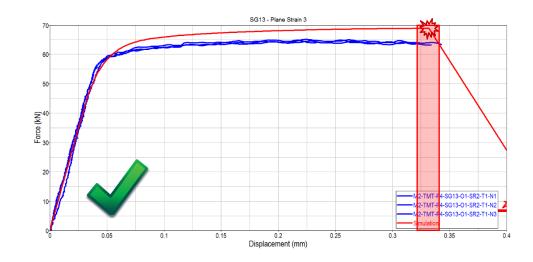


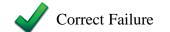


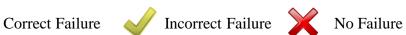


## **Plane Strain**



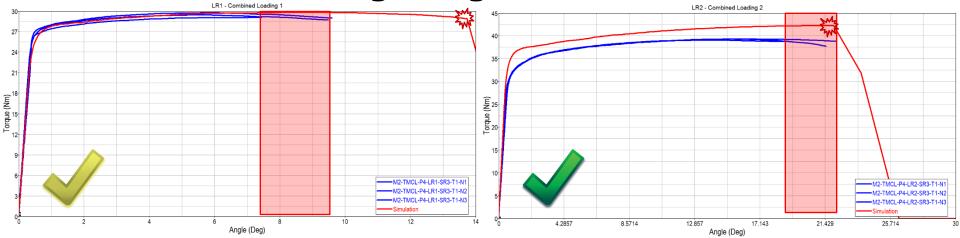


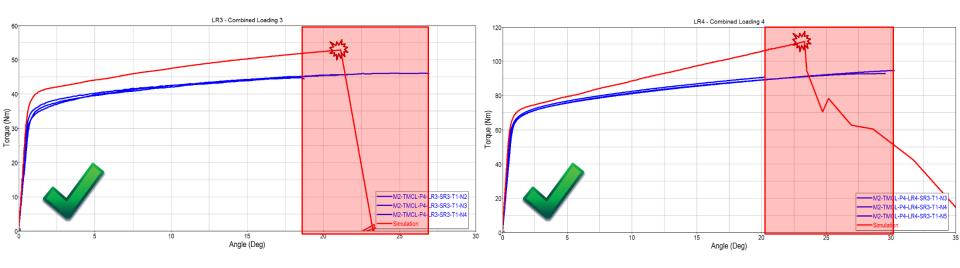


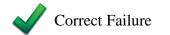




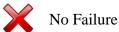
# **Combined Loading - Angle**



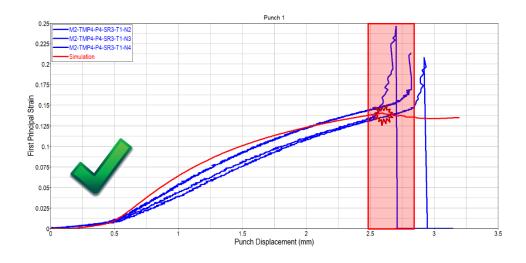


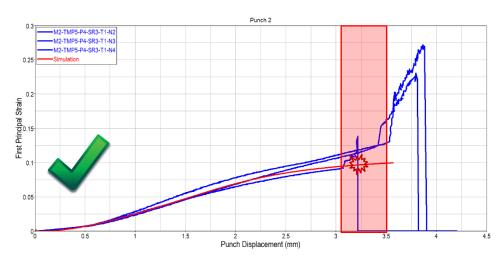


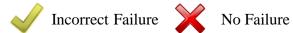


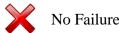


## **Punch Tests**

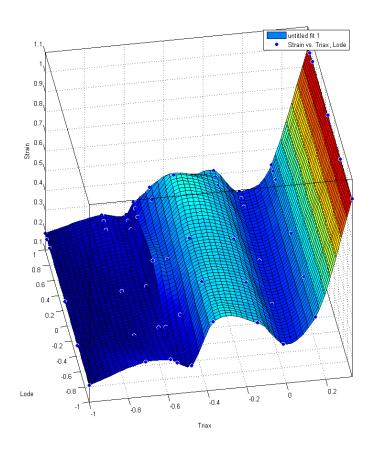


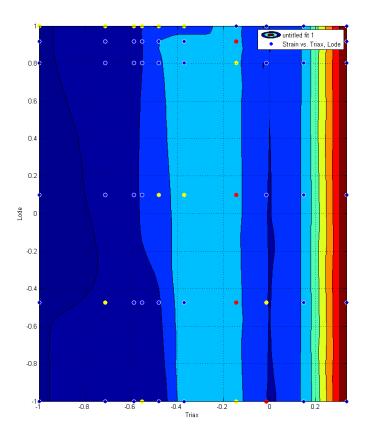






## **Final Failure Surface**

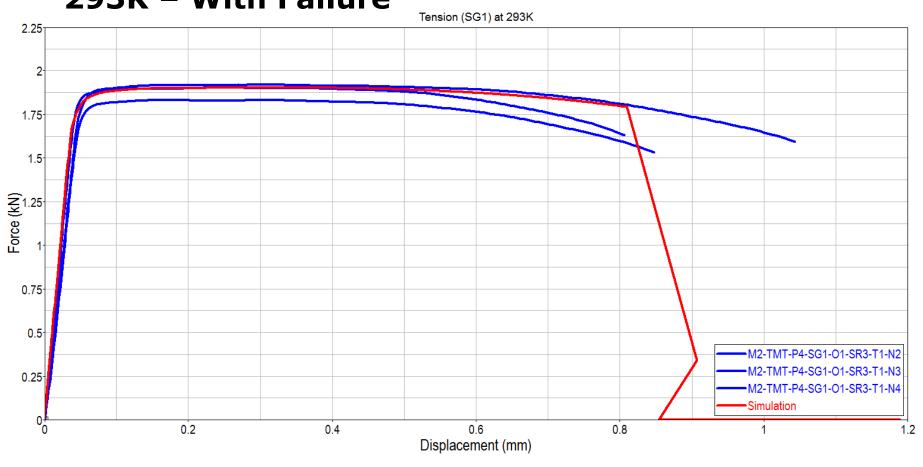




# **Temperature dependency**

- Generate scaling factor for failure surface based on temperature simulation results
- Temperature tests were quasi-static, so we assume no rate effects, they were also isothermal, so we assume no adiabatic (heat) effects
- •Therefore every temperature test is simulated at an arbitrary speed (1000 times higher then reality) using a SINGLE hardening curve specified as the rate dependent 'table'
- •This allows to determine a failure strain for every temperature in the usual way by matching the displacement at failure
- •From the failure strain for every temperature we derive a scaling function for the failure surface, thus we assume that the temperature effects on the failure surface are independent of the state of stress
- •This is 'optimistic' but compatible with the multiplicative split approach used in the Johnson-Cook type laws



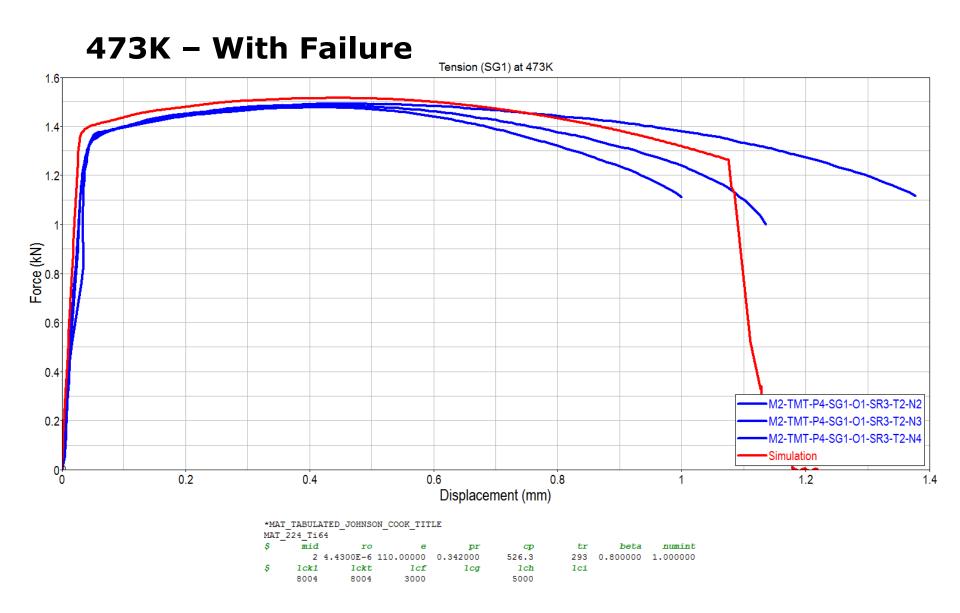


\*MAT\_TABULATED\_JOHNSON\_COOK\_TITLE

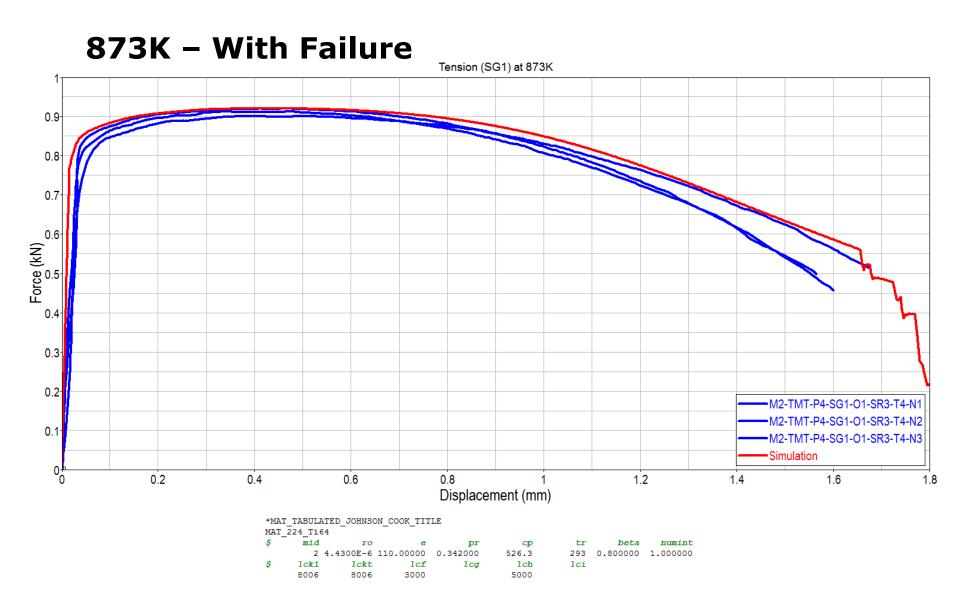
MAT\_224\_Ti64

\$ mid ro e pr cp tr beta numint
 2 4.4300E-6 110.00000 0.342000 526.3 293 0.800000 1.000000

\$ lck1 lckt lcf lcg lch lci
 8003 8003 3000 5000







# Temperature dependency of failure

Created a temperature scaling curve for the failure surface

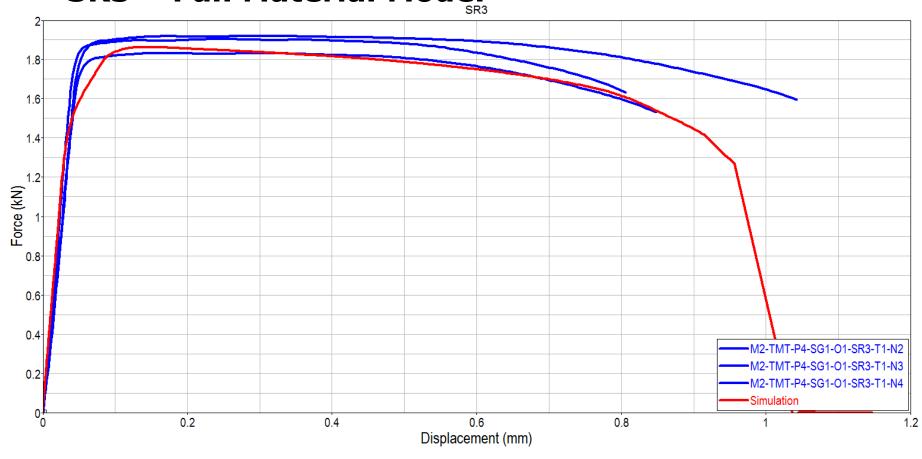
```
Failure - Temperature Curve
*DEFINE CURVE
   LCID
         SIDR
               SFA
                     SFO
                          OFFA
                                OFFO
                                     DATYP
   5000
s
          A1
                      01
        293.00
                    1.000
        473.15
                    1.950
        673.15
                    1.950
        873.15
                    2.950
       1877.00
                    2.950
```

Capped at 873 K

# **Strain Rate dependency**

- •We now simulate the rate dependent tests SR3, SR4 and SR5 at real life velocities using the full input: rate dependent table, temperature dependent table, failure surface, temperature scaling function and strain rate scaling function
- SR3 is 1/s, SR4 is 600/s and SR5 is 1500/s, in dynamic experiments we have temperature as well as strain rate effects
- •Some tuning of the strain rate scaling function is necessary to obtain good failure response in all 3 cases

## **SR3 – Full Material Model**

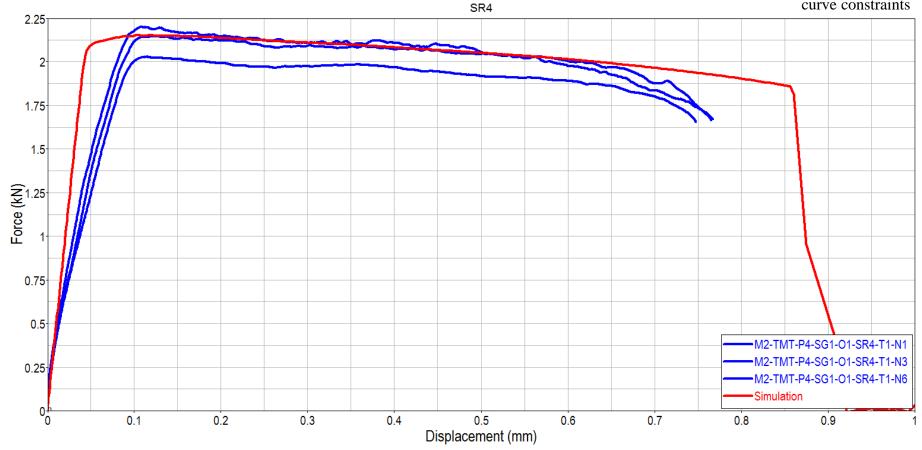


\*MAT\_TABULATED\_JOHNSON\_COOK\_TITLE
MAT\_224\_Ti64
\$ mid ro e pr
2 4 4300F=6 110 00000 0 342000

\$ mid ro e pr cp tr beta numint
2 4.4300E-6 110.00000 0.342000 526.3 293 0.800000 1.000000
\$ lck1 lckt lcf lcg lch lci
1000 2000 3000 4000 5000



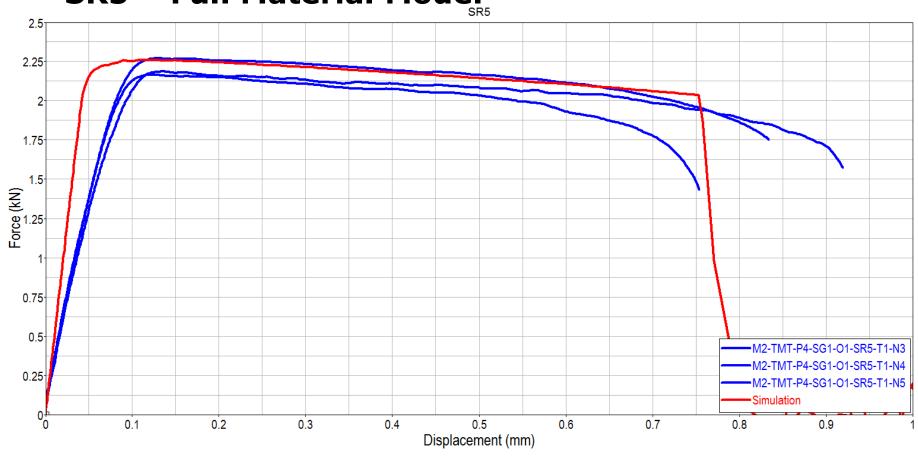
Failure displacement due to non-monotonic curve constraints



\*MAT\_TABULATED\_JOHNSON\_COOK\_TITLE MAT\_224\_Ti64

MAI	224 116	4						
\$	mid	ro	е	pr	сp	tr	beta	numint
	2	4.4300E-6	110.00000	0.342000	526.3	293	0.800000	1.000000
\$	1ck1	lckt	lcf	1cg	1ch	lci		
	1000	2000	3000	4000	5000			

## **SR5 – Full Material Model**

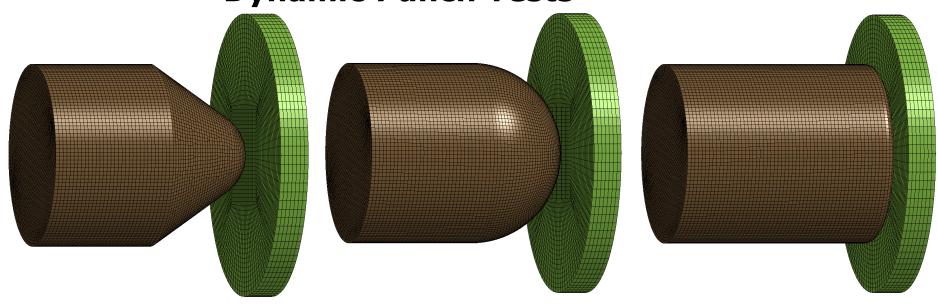


\*MAT TABULATED JOHNSON COOK TITLE MAT 224 Ti64 2 4.4300E-6 110.00000 0.342000 526.3 293 0.800000 1.000000 1ck1 lckt 1cf 1cg 1ch lci 1000 2000 3000 5000 4000

# **Strain Rate Failure Curve Adjustments**

\$ \$\$\$\$		\$\$\$\$\$\$\$\$\$\$	- Strain R	ate Curve		Ş	
\$	LCID	SIDR	SFA	SFO	OFFA	OFFO	DATYP
	4000						
<i>\$</i>		A1		01			
	0	.000010		1.000			
	0	.001000		0.750			
	2	.350000		0.370			
	6	.610000		0.370			
\$\$\$\$	\$\$\$\$\$\$\$\$\$\$	\$\$\$\$\$\$\$\$\$\$\$	\$\$\$\$\$\$\$\$\$\$	\$\$\$\$\$\$\$\$\$\$	\$\$\$\$\$\$\$\$\$\$	\$\$\$\$	
\$		Failure -	- Temperat	ure Curve		\$	
\$\$\$\$	\$\$\$\$\$\$\$\$\$\$	\$\$\$\$\$\$\$\$\$\$\$	\$\$\$\$\$\$\$\$\$\$	\$\$\$\$\$\$\$\$\$\$	\$\$\$\$\$\$\$\$\$\$	\$\$\$\$	
*DEE	INE CURVE	ı					
\$	<u>LC</u> ID	SIDR	SFA	SFO	OFFA	OFFO	DATYP
	5000						
\$		A1		01			
		293.00		1.000			
		473.15		1.950			
		673.15		1.950			
		873.15		2.950			



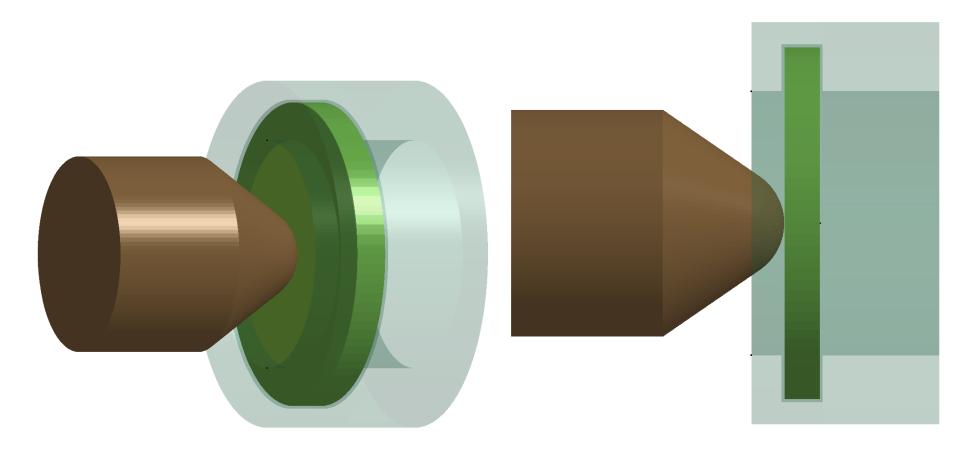


#### Full Material Model:

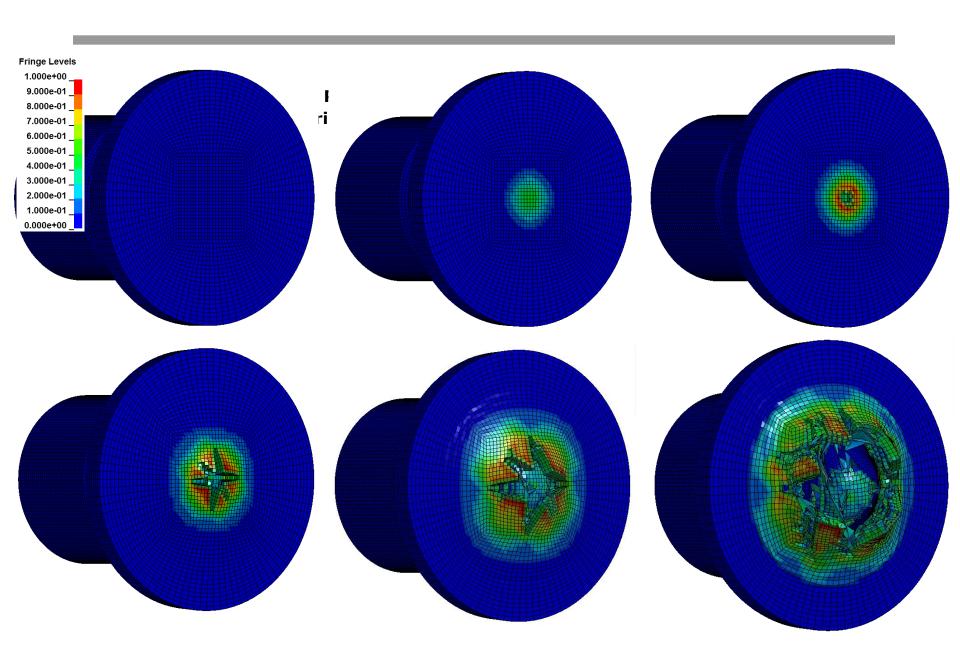
```
*MAT TABULATED JOHNSON COOK TITLE
MAT 224 Ti64
       mid
                                                                    beta
                                                                            numint
                  ro
                                       \mathbf{pr}
                                                  CP
         2 4.4300E-6 110.00000 0.342000
                                              526.3
                                                           293
                                                                0.800000
                                                                          1.000000
      1ck1
                1ckt
                          1cf
                                                1ch
                                                           1ci
                                      1cg
      1000
                2000
                          3000
                                     4000
                                               5000
```

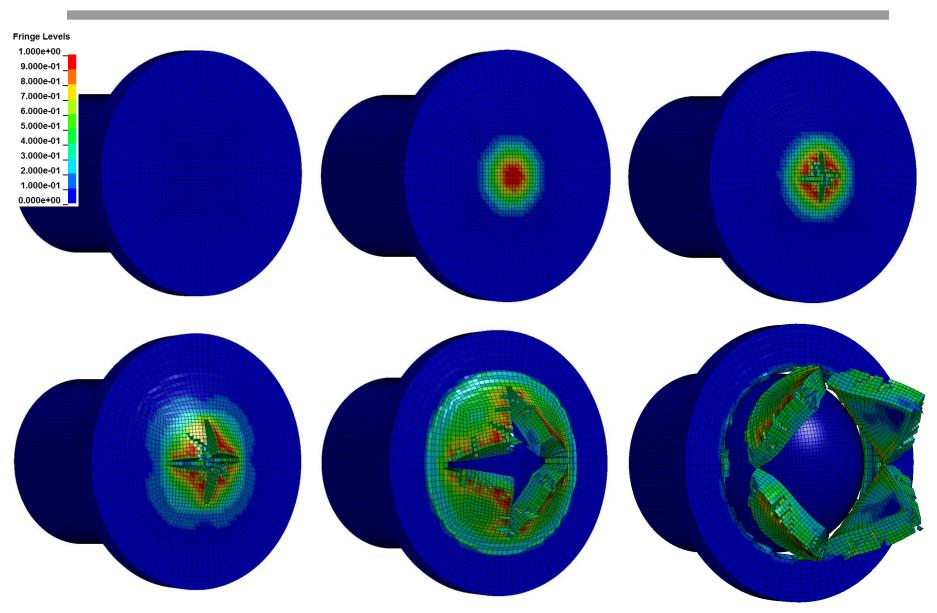
# **Boundary Conditions**

The adapter and clamp assembly was modeled

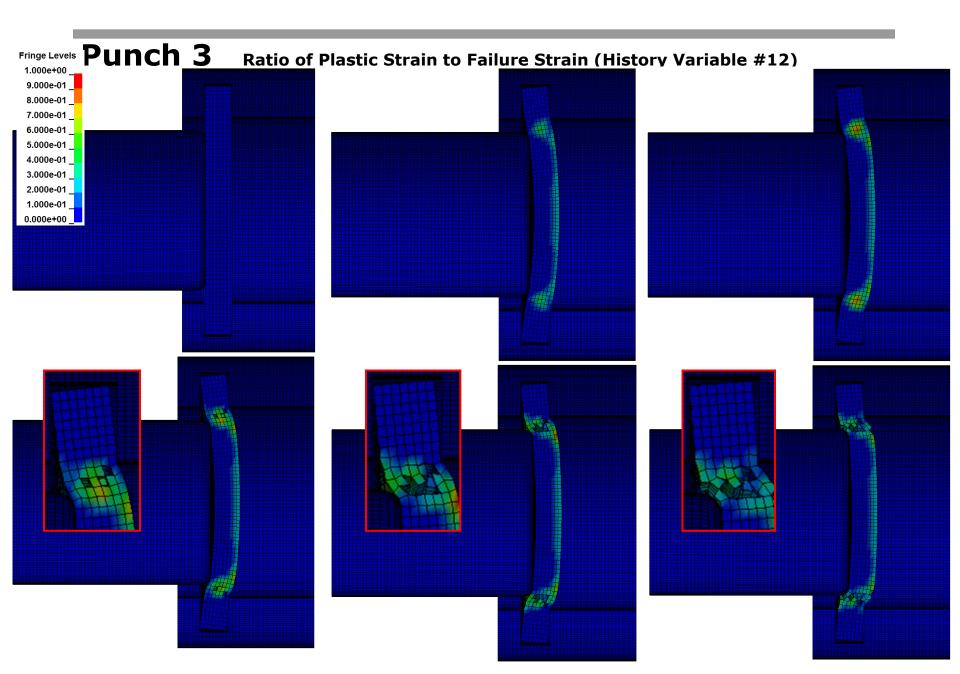


#### **Numerical Simulation of Dynamic Failure : ballistic tests on Titanium-6Al-4V**

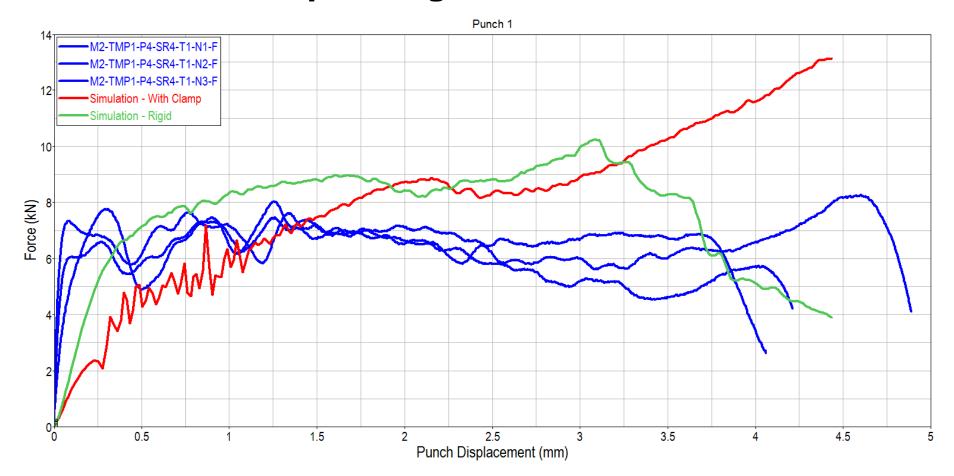




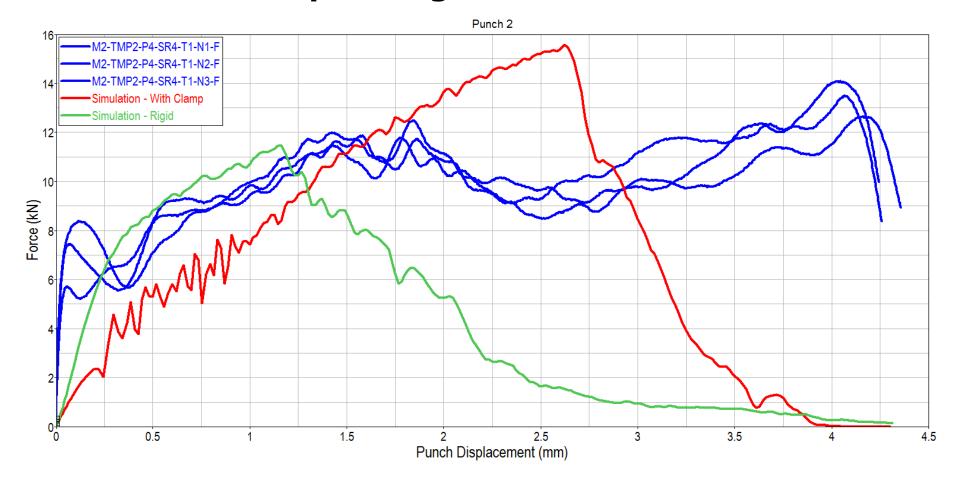
Punch 2 Ratio of Plastic Strain to Failure Strain (History Variable #12)



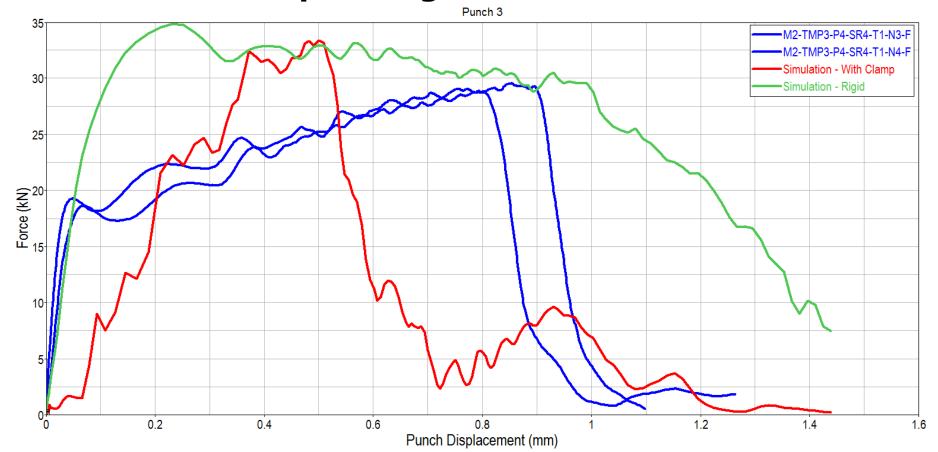
# Punch 1 Clamp vs. Rigid



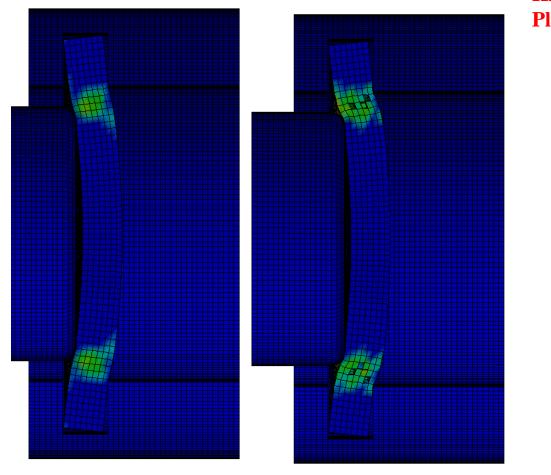
# Punch 2 Clamp vs. Rigid



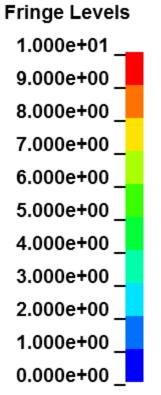
# Punch 3 Clamp vs. Rigid



- Strain rates range from 5 1/ms to 10 1/ms
- This corresponds to curves 1009 and 1010

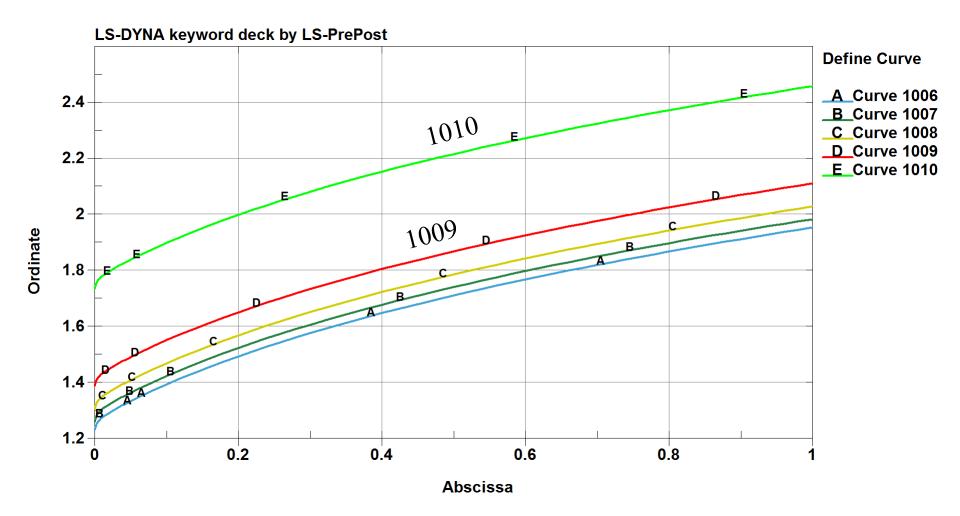


# History Variable #5: Plastic Strain Rate



**Strain Rates for Punch 3** 

## **Yield Curves**



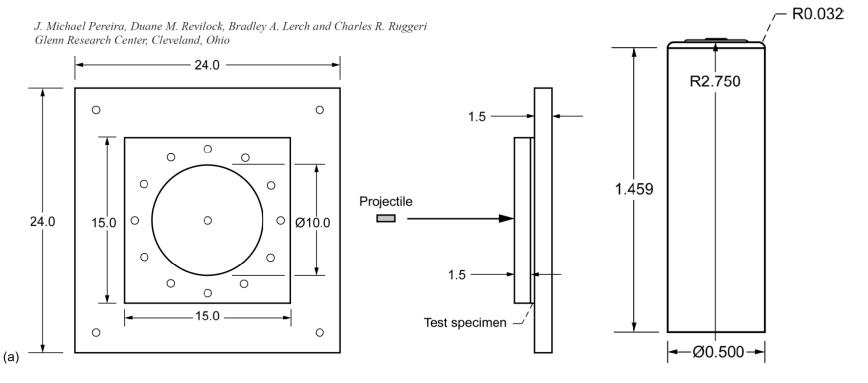
### **NASA Ballistic Test Series**

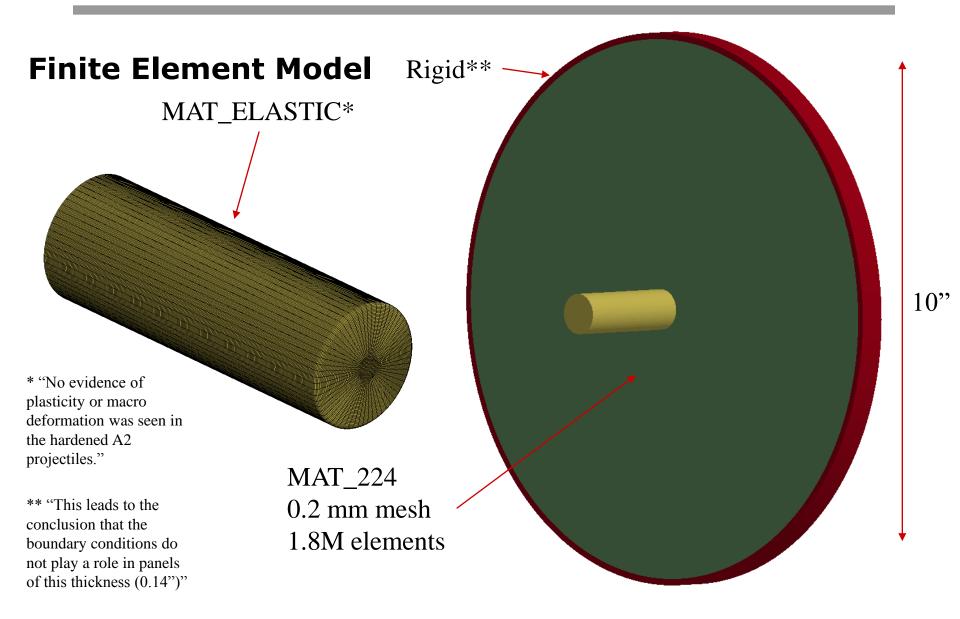
NASA/TM-2013-217869

DOT/FAA/TC-12/58



# Impact Testing of Aluminum 2024 and Titanium 6Al-4V for Material Model Development





#### **Numerical Simulation of Dynamic Failure : ballistic tests on Titanium-6AI-4V**

Test	Projectile mass, gram	Projectile impact velocity, ft/sec	Projectile exit velocity, ft/sec
DB177	126.3	896	475
DB178	126.4	865	424
DB179	126.2	713	241
DB180	126.2	646	152
DB182	126.4	527	0
DB184	126.3	581	0
DB185	126.2	597	0
DB186	126.4	578	0
DB192	126.3	630	0
DB193	126.3	629	104
DB195	126.3	616	0

<sup>\*</sup> Assume zero degree impact angle

## **Triaxialities in ballistic impact tests**

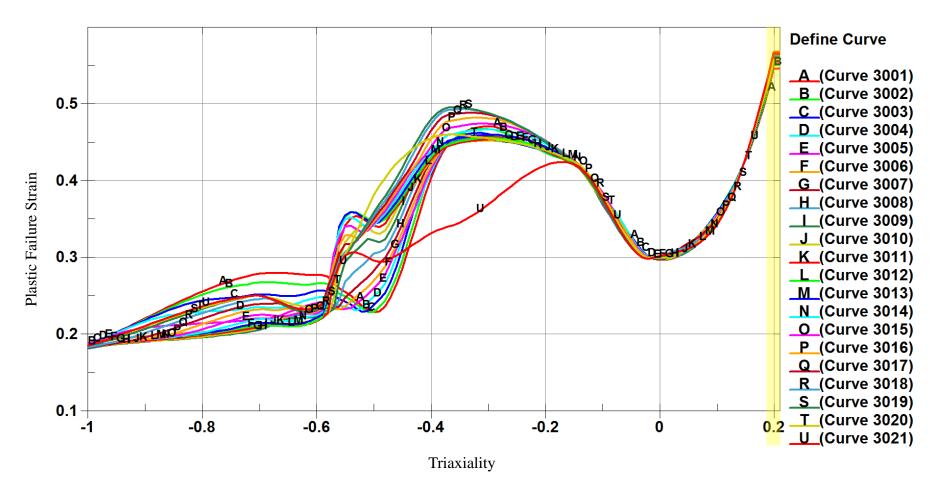
 Triaxialities in and around the ASB were between 0.2 and 5 showing a very high hydrostatic pressure contents

$$\mathbf{\sigma} = \begin{pmatrix} -p & 0 & 0 \\ 0 & -p & 0 \\ 0 & 0 & -p \end{pmatrix} + \begin{pmatrix} \tau & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & -\tau \end{pmatrix} \quad p > 0 \text{ and } \tau > 0 \quad t = \frac{p}{\tau\sqrt{3}} > 0$$

if  $p > \tau$  all stress components < 0

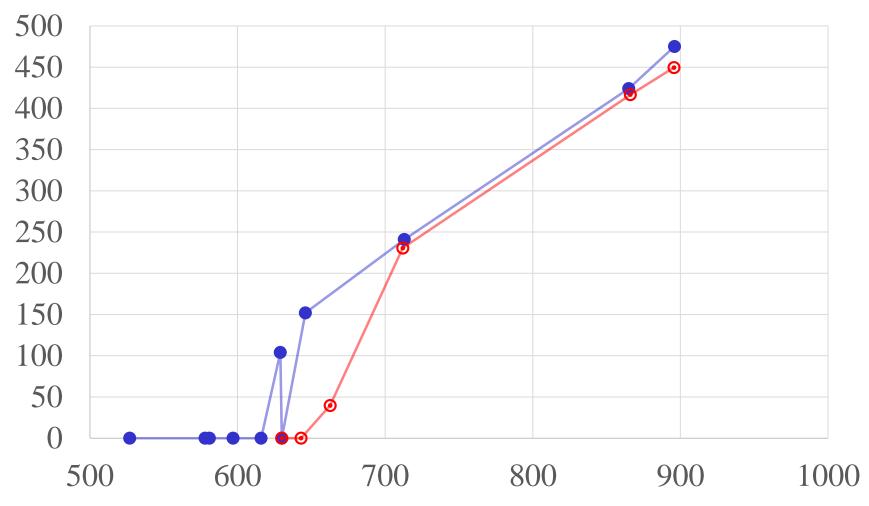
- Our failure surface had test data for triaxialities up to 0.147 (combined loading test LR4)
- The decisive failure in the ballistic impact tests (ASB) was completely in a dark region of the failure surface (triaxialities >0.2)
- An update of the failure surface was needed
- The assumption that no failure occurs when all 3 principal stress components are negative (triaxiality > 0.3333) is FALSE

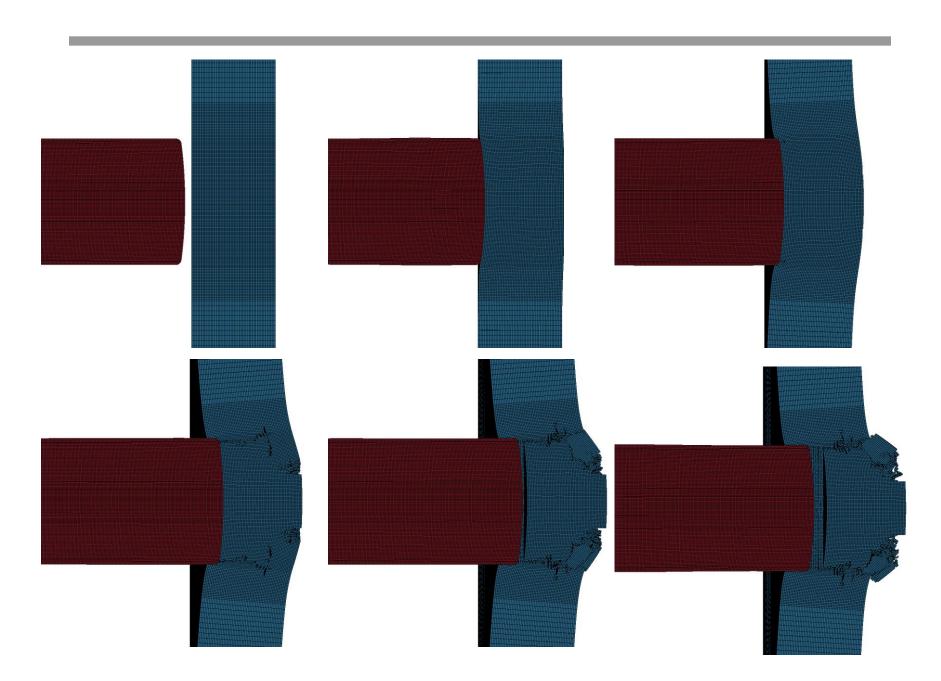
## **Adjustments to Failure Surface**

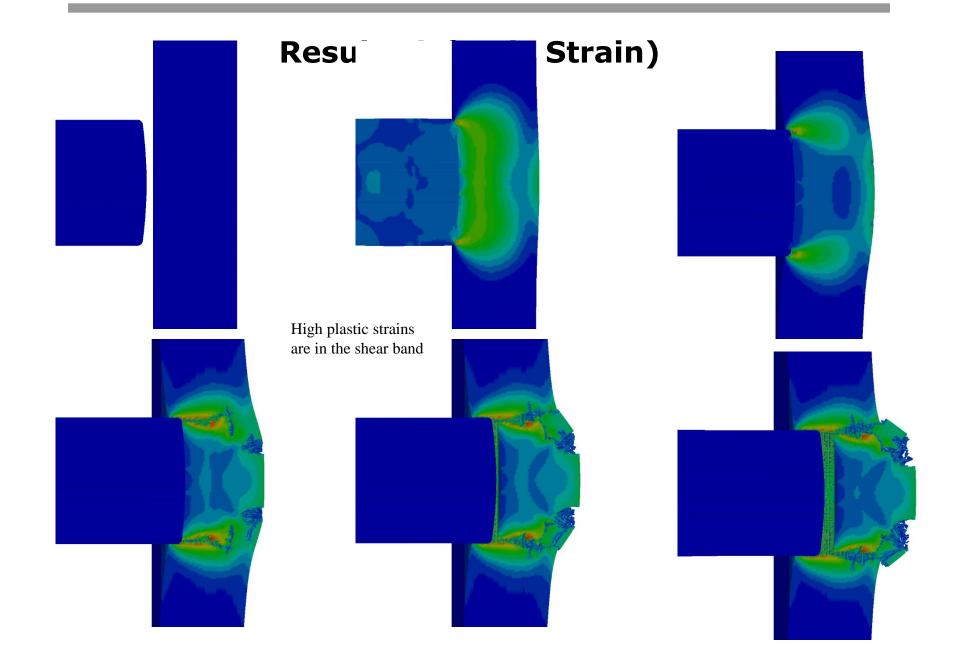


Introduction of a plateau in the failure surface at 60% for triaxialities > 0.2, NO previous tests are affected

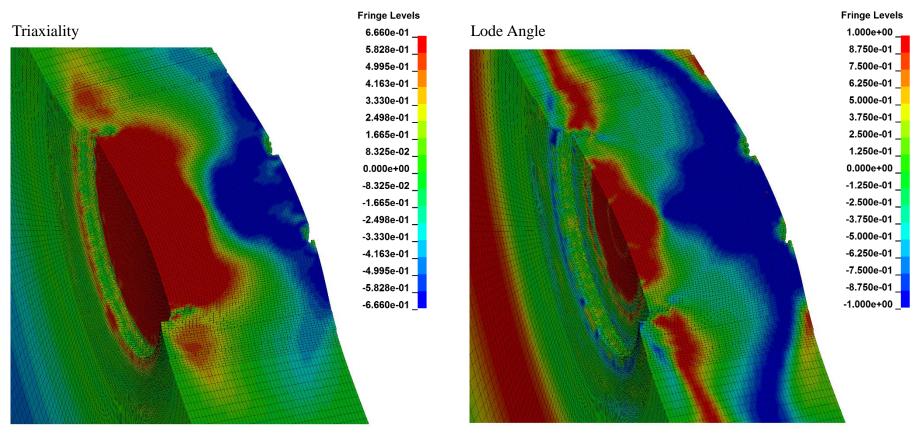
### **Simulation Results**







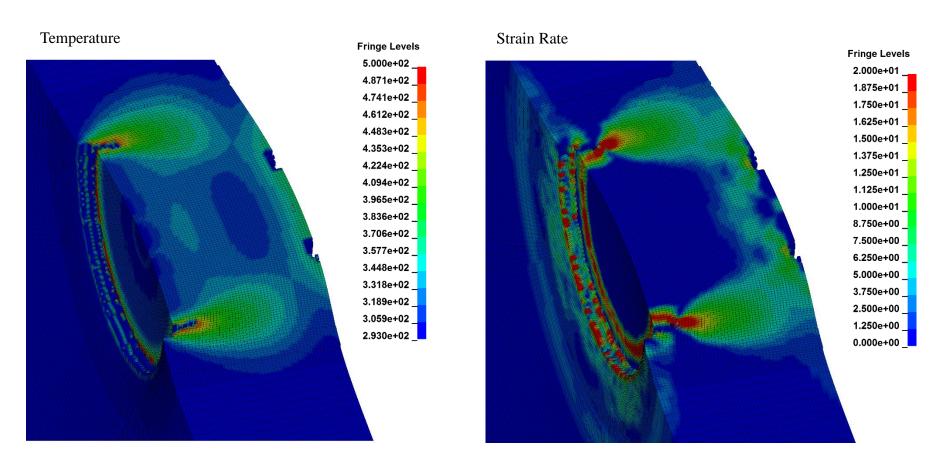
## **Results (Triaxiality and Lode)**



Triaxialities exceed 0.666

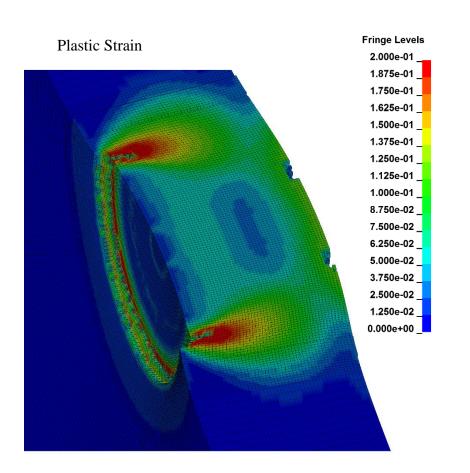
Lode parameters in the ASB between -0.5 and -1

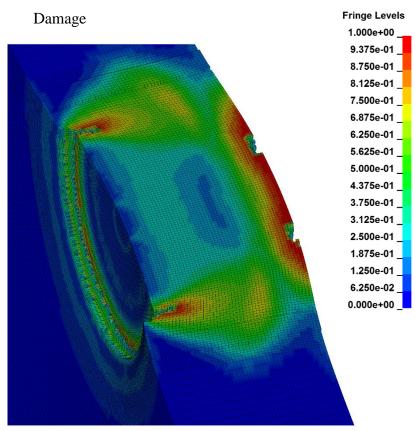
# **Results (Temperature and Strain Rate)**



Temperature rise of around 200C in the shear band, negligable in the spall plane Strain rate values of 20000/s and higher

# **Results (Plastic Strain and Damage)**

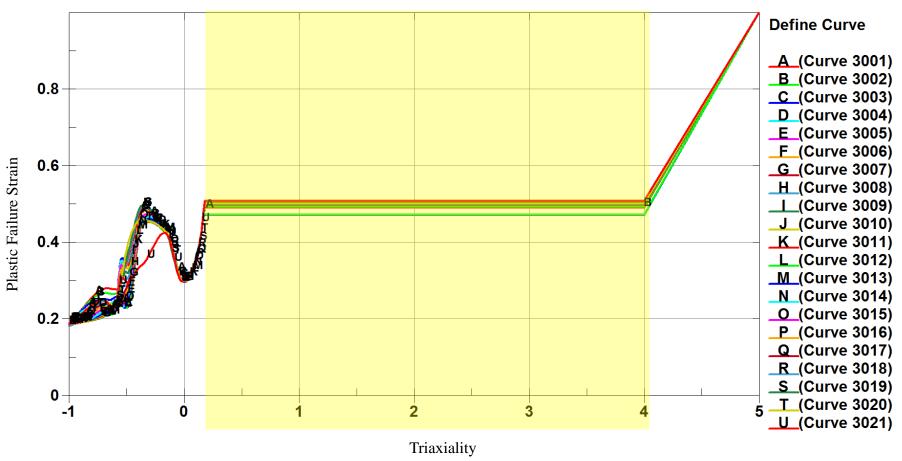




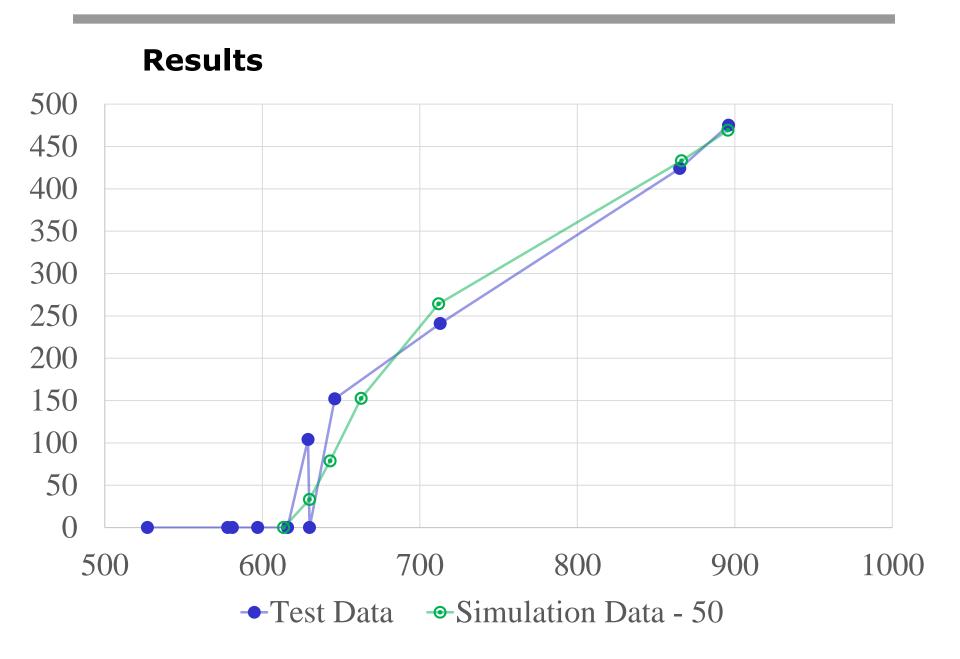
#### **Discussion of results**

- Some spalling occurs in the simulation, more then observed in the test, this could be due to the ASB forming somewhat too late
- According to Rittel: 'the influence of the hydrostatic pressure on the formation as ASB is still an open question'
- Our shearband can propagate too slow due to the 0.2mm mesh size, according to Meyers the width of shear bands in Ti-6-4 is between 2 and 20 micron, according to Borvik the mesh size should not exceed the half width of the shear band

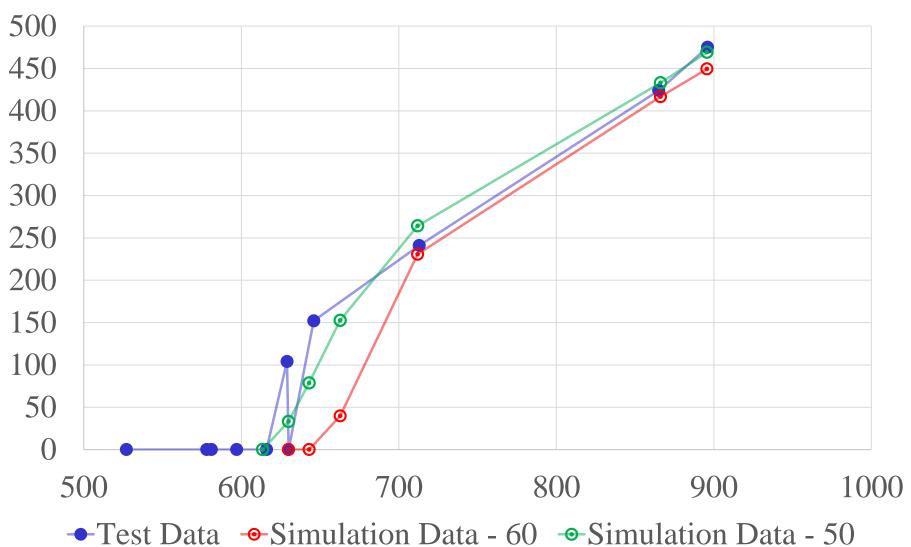
#### **Adjustments to the Failure Surface**



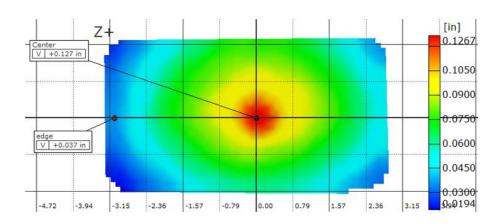
Shift of the plateau from 60% to 55%

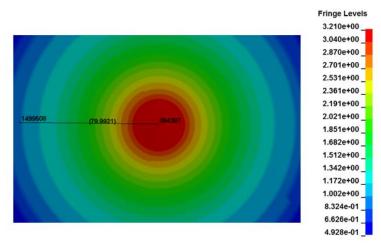






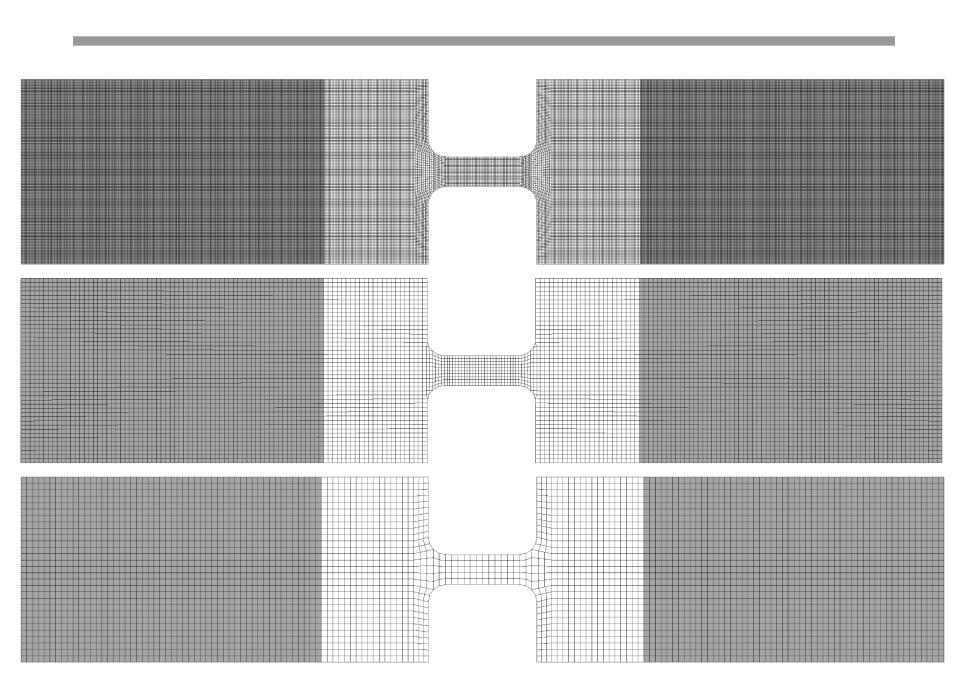
### Comparison of displacements for DB182 (527 ft/sec)



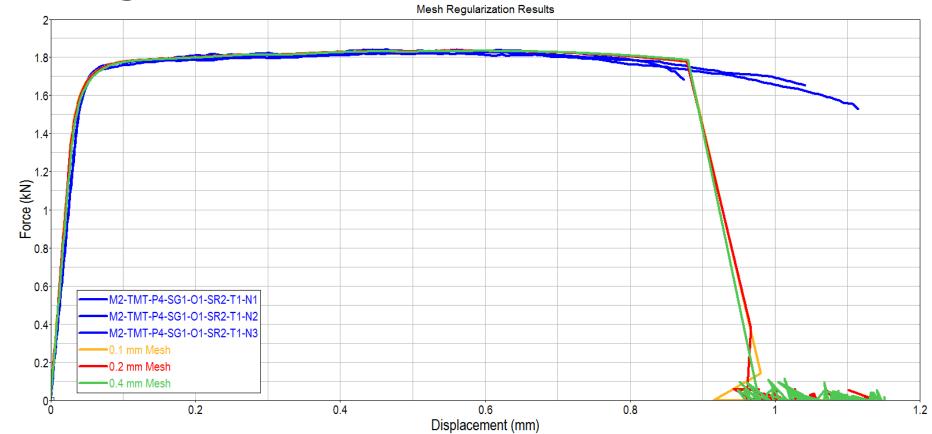


#### **Regularization Curve**

- A regularization curve was developed using three mesh sizes
  - •0.1 mm
  - •0.2 mm
  - •0.4 mm
- Each specimen was simulated using the entire Ti-6-4 material model
   Including failure regularization curve
- The regularization curve was adjusted with each iteration until the failure for each specimen occurred at the same displacement



### **Regularization Results**

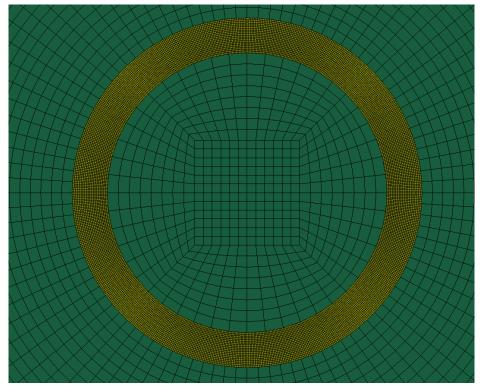


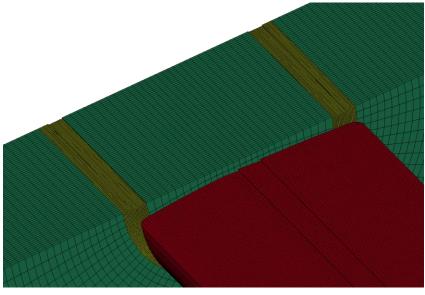
#### **Final Regularization Curve**

```
Failure - Regularization Curve
*DEFINE CURVE
$#
  lcid sidr sfa
                 sfo
                       offa offo
                                dattyp
   6000
          1.0
                  1.0
                       0.0 0.0
     ABSCISSA
                ORDINATE
        0.00
                 1.038
        0.10
                 1.038
        0.20
                 1.000
        0.40
                 0.912
        0.50
                 0.912
```

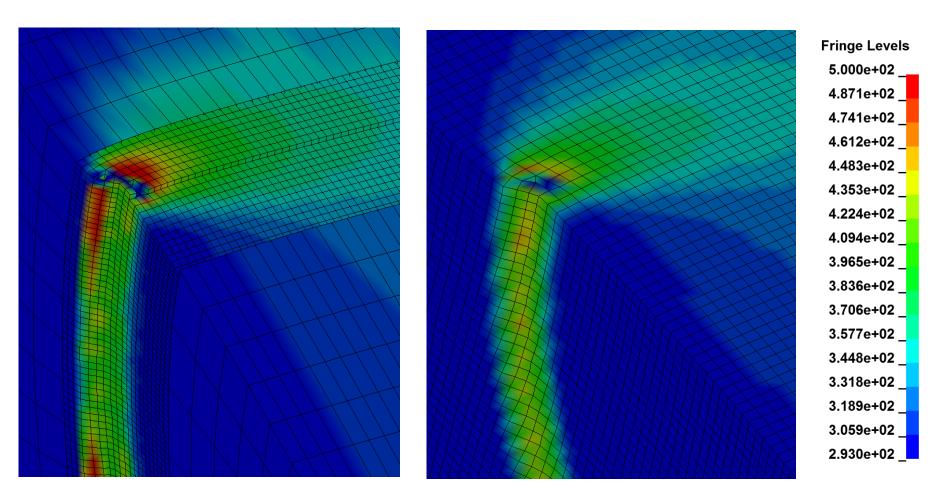
#### NASA Ballistic Test (0.1 mm mesh)

- NASA Ballistic test plate was re-meshed with 0.1 mm elements in the area of the adiabatic shear band
  - •0.1 mm elements
  - •Connected with \*CONTACT\_TIED\_SURFACE\_TO\_SURFACE
  - •217 m/s impact velocity





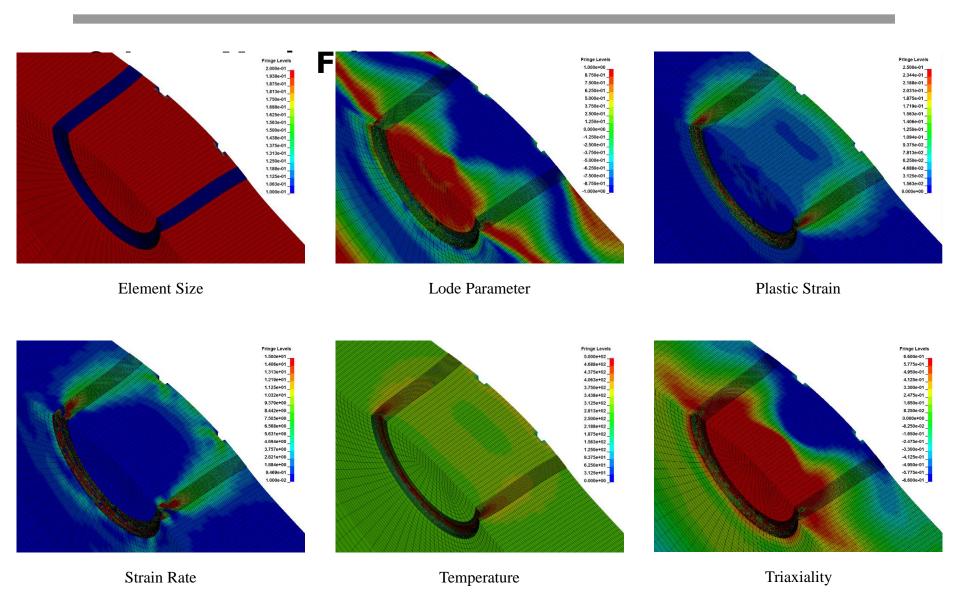
## **Temperature Comparison (0.02ms)**

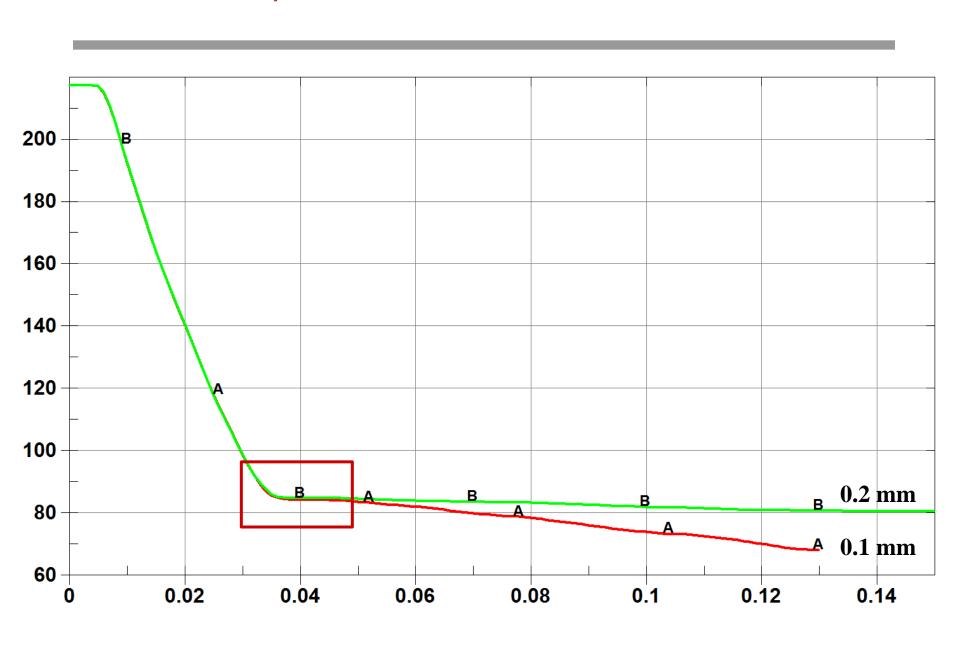


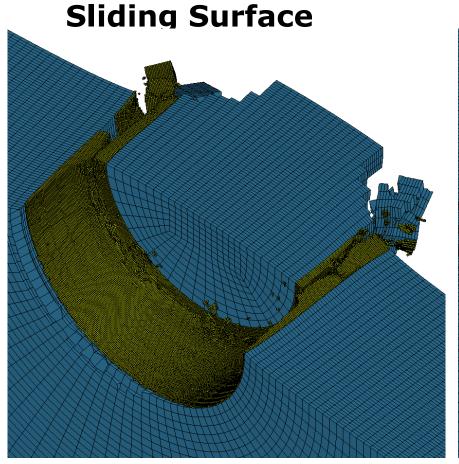
Max Temperature = 634 K

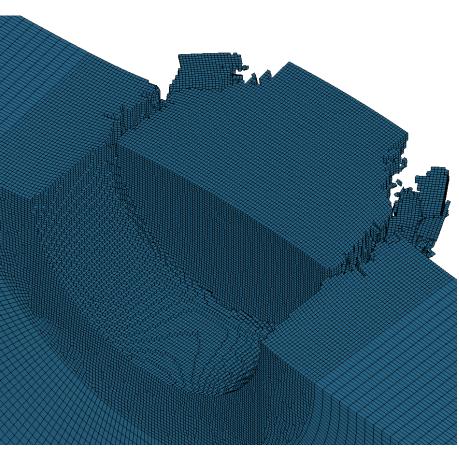
Max Temperature = 569 K

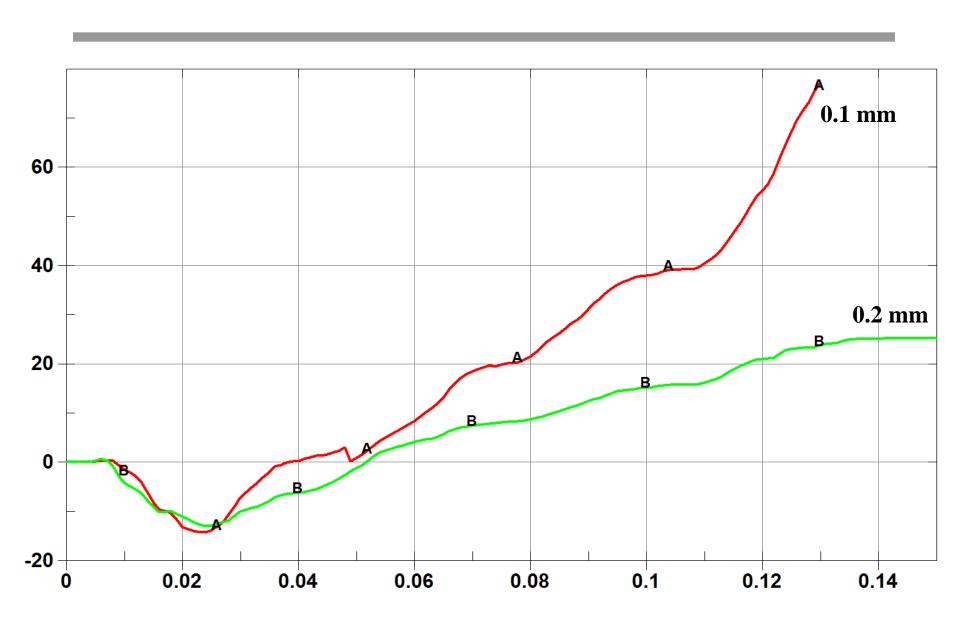
#### Numerical Simulation of Dynamic Failure: ballistic tests on Titanium-6Al-4V











#### **Conclusions**

- The mechanisms of plastic instability and failure can be very different under static and dynamic loads
- Whereas regularization is often not needed to model shear failure under quasistatic, isothermal conditions, the situation may be different under high dynamic loads
- In the present study the mesh size was much larger then the width of the physical ASB, still a shear failure occurred in the simulation and realistic exit velocities could be obtained
- It is wrong to assume that no failure will occur in the region of stress space where triaxialities have large positive values, more attention will be given to this part of the failure surface in the future