



# **Gestamp** BIW



### Dynamore Forum 2012

## **Cross Car Beam Multi Optimization**

Authors: Francesc Volart Sergio Faria





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## **Gestamp Corporation**



- Body-in-White, Chassis and Mechanisms
- Wide range of technologies.

**Gestamp** BIW

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Leadership



## **II. Gestamp Automotive Portfolio**





## 2. The Project

#### I. Context

#### **Cross Car Beam Assembly**



Cross Car Beam is a support that usually holds all Instrument Panel including HVAC System, Knee Airbags, **Steering Column**, Radio and many other components.

#### **Previous Optimization Experience**

- Simple morphing shapes (beads, flanges)
- Material properties
- Modal and static analysis

#### New challenges of this New Project

- Complex morphing shapes including remeshing
- · Components position displacement
- Welding projection after shape modification
- Include both static and dynamic analysis

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## 2. The Project



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#### I. Optimization Flow-Chart



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#### I. Optimization Flow-Chart

#### Morphing process





II. First Optimization Phase (NVH) – DOE studies

#### Morphing examples







II. First Optimization Phase (NVH) – DOE studies

#### Design variables studied

Example driver's tube morphing



Shape X Right



Shape Z Right



Shape 2 Kight

Shape X Left







#### **Correlation matrix**



VARIABLES		
Side bracket position	Passenger's tube shape Z Right	
Passenger's tube position X	Driver's tube shape X Left	
Passenger's tube position Z	Driver's tube shape Z Left	
Driver's tube position X	Driver's tube shape X Right	
Driver's tube position Z	Driver's tube shape Z Right	
Column position X	Driver's tube shape X Middle	
Passenger's tube size	Driver's tube shape Z Middle	
Passenger's tube size Left	Passenger's tube shape X	
Passenger's tube size Right	Passenger's tube shape Z	
Driver's tube size	Driver's tube shape X	
Driver's tube size Left	Driver's tube shape Z	
Driver's tube size Right	Thickness properties	
Passenger's tube shape X Left	Materials properties	
Passenger's tube shape Z Left		
Passenger's tube shape X Right		







II. First Optimization Phase (NVH) – DOE studies

#### Analysis Results - Metamodels



Steering wheel stiffness

**Deflection test** 

Metamodels were used to redefine the range of the design variables.

For example the driver's tube diameter range changed from [40-70]mm to [55-70]mm

#### **First Phase Results**

- After this first optimization phase the amount of design variables was reduced from 35 to 10
- 2. The design space was also reduced changing the design variables range to the place of the best response results.



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#### III. Second Optimization Phase (NVH and Crash)

#### **Design Variables**

Some design variables were added for frontal crash optimization.

In crash analysis materials stress strain curves were used as design variables as well.





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III. Second Optimization Phase (NVH and Crash)

#### **DOE Study**

DOE study was used to analyse and have more information about the new variables.



History of section force on driver's tube - Side impact

#### Variables and responses after DOE

VARIABLES	RANGE
Passenger's tube position X	[0 – 50] mm
Passenger's tube size	[25 – 40] mm
Driver's tube position X	[0 – 60] mm
Driver's tube size	[40 – 50] mm
Firewall bracket shape top	[0 – 7] mm
Firewall bracket shape bottom	[0 – 7] mm
Firewall bracket shape side	[0 – 7] mm
Firewall bracket shape width	[-10 – 15] mm
Passenger's tube thickness	[1, 1.5] mm
Driver's tube thickness	[1.5, 2.0, 2.4] mm
Tunnel leg thickness	[1.0, 1.5] mm
Passenger's tube material	[1, 2]
Driver's tube material	[1, 2]
Tunnel leg material	[1, 2]

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RESPONSES	TYPE	SIMULATION
1st frequency	Constraint	Modal analysis
2nd frequency	Constraint	Modal analysis
Weight	Objective	-
Deflection	Constraint	Deflection test
Steering wheel displ.1	Constraint	Steering stiffness
Steering wheel intrusion	Constraint	Frontal crash
Reaction force	Constraint	Side impact

Design variables and its ranges after DOE studies





III. Second Optimization Phase (NVH and Crash)

#### Calculation time distribution (minutes)



- Modal analysis
- Deflection test
- Steering wheel stiffness
- Frontal crash
- Side impact

#### **Optimization used**

In order to achieve the best variables configuration and reduce the number of experiments it's used Sequential Response Surface Method (SRSM).



- LS-OPT configuration:
  - Metamodel:
    - Polynomial Quadratic
  - Point selection:
    - Space Filling default 181 points

#### **Optimization history (weight)**



The optimization calculated about 20 iterations and more than 220 different configurations. Total optimization time took 3 days (4 CPU's).





### 4. Results and conclusions

#### I. Optimization results (NVH and Crash)

#### **NVH results**





## 4. Results and conclusions

#### I. Optimization results (NVH and Crash)

#### Crash results



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## 4. Results and conclusions

#### II. Conclusions

In the beginning of this project the packaging was very restrictive, the position of the components could not move too much as in a **RFQ phase** and the final optimization **did not achieved very good results**.

Therefore we decided to ignore some parts of the packaging as it were a **concept phase**. Increasing the movement of the components we could check for possible positions which we had been never tried before and see they could be a good solution design.

The final weight reduction is about 18%. (5,70 kg to 4,67 kg) achieving all targets:

- · Modal analysis,
- Deflection test
- Steering column stiffness
- Frontal crash
- Side impact

LS-OPT is a useful and great tool to coordinate different kind of simulations and analyse the results.

However, working with large number of variables could carry some difficulties to manage all together.

In this project a great part of the time was dedicated in the FE-Model parameterization, welding scripting and learning LS-Opt features which we hope to reduce this time for next projects.





