Recent Developments of LS-DYNA® in Stamping Simulation

Xinhai Zhu

November, 2009



Outline

Implicit method
Material work-hardening
Some new keywords
Surface low detections
Conclusions



Improvements in Implicit Method



Implicit Method

Implicit method has been gaining more popularities in sheet stamping simulation

- Initial application was mainly limited to springback predictions
- Gravity loading simulation has been proven to be robust with implicit method
- Implicit method has also shown great potentials in flanging simulation.



- Binder wrapping characteristics
 - Large blank movements, large dynamic effect
 - Small plastic deformation
 - Element is relatively coarse, and the number of element is small
 - Implicit method might be the preferred one
- Recent developments
 - New implicit contact
 - New features to help convergence
 - An extensive parameter studies
 - The default parameters are suitable for most of the binder wrapping process
 - It is easy to use
 - User in-dependent results can be obtained



- The proper step size for a typical binder wrapping process
 - The new algorithm allow large time step. While the old time will limit the time step size
 - Too small time step will require many time steps
 - Too large time step will require more iterations, sometimes, the result is not good





Too small time step might have convergence problem





Reasonable time step should make sure that most of the rigid body motions are constrained





- A typical time step selection
 - 10 time steps are used
 - It took 12 minutes to finish with ONE CPU





- Sinder and Gravity Loading are combined into one simulation
 - In the old method, gravity loading has to be done separately
 - It allow mesh refinement during each step
 - It allowe quide pipe





After Gravity Loading, blank elements are refined



- Final geometry: all the buckling mode has been correctly simulated
 - It took 90 minutes to finish





Implicit Method: Other Applications

- Initially developed for metal stamping simulations that involve gravity loading of blanks on dies.
- Now extended as a general capability
 - Serial, SMP and MPP implementation
- Robust contact treatment
- Vehicle does not need to be supported to eliminate rigid body modes
- Elimination of loose parts not required
- Reduces model preparation time dramatically over traditional implicit method.



- Sometimes, springback predictions are not accurate
 - Scan data can be used to compensate the springback deviations
- Procedure in using scan data
 - Perform a forming simulation, and obtained the deformed part
 - Assume the scan data as rigid tools
 - Assume the deformed as a deformable
 - Apply internal pressure to the deformed part
 - After push, the blank geometry can be used as sprung shape
 - Use the same procedure as before and compensate the rigid



- How to push the deformed part to the scanned geometry?
 - The pressure is applied by using load mask
 - All the blank element will have a normal pressure
 - The pressure is applied to the opposite of element normal directions
 - Blank normal has to be checked
 - For most of the situation, the internal pressure can be in the range of 20 ~ 30 MPA
 - Use implicit method
 - Use the keyword: *CONTROL_IMPLICIT_FORMING
 - One step pushing is used
 - The CPU time is small (usually can be done within a few minutes)



Benchmark Study:

- Number of element: 41,457
- CPU cost: 7 minutes and 39 seconds





Implicit Method Compensation Based on Scan Data Problems might happen during the pushing: No support for the boundary elements

MPLICIT LS-DYNA iteration plot databas The * 0.100 MPLICIT LS-DYNA iteration plot databas The * 0.12 MPLICIT LS-DYNA iteration plot databas The * 0.12 The *



Over-bend in the boundary areas

Change the boundary elements to a different part

Avoid applying pressure to the boundary element



- After first tryout, the parts exhibited significant springback issues.
- Part could not even be placed on the checking fixture without extreme hand-working.
- Part was bowed throughout the length of the channel, twisted, there were bulges adjacent to the deeper areas of form, form depth in the deeper areas was incorrect, and the stepped flanges were crowning.







- There is a dramatic change.
- A significant amount of springback has been removed
- With adjustments to magnification, an even more effective compensated shape could have been created.





Livermore Software Technology Corporation

Courtesy of Matt Clarke (Continental Tool And Die)

Material kinematic hardening Yoshida model Chaboche's model



Springback Prediction for HSS

Some parts are sensitive to stress noise
More accurate stress calculation is important





Twisting mode corresponding to the lowest frequency (17.97) And is far smaller than the next one: 54.7 Accordingly, small stress noise might excite the wrong twisting mode

- Recent researches found that Yoshida nonlinear kinematic hardening is the preferred one.
 - It can describe the softening effect from reverse loading
 - It can accurately represent the stress-strain curve from the cyclic loa



Key points of Yoshida's theory:

- Yield surface does not change in size
- Center of yield surface moves with deformation
- Bounding surface change both in size and location

$$\boldsymbol{\alpha}_{*} = \boldsymbol{\alpha} - \boldsymbol{\beta}.$$

$$\overset{\circ}{\boldsymbol{\alpha}_{*}} = C \left[\left(\frac{a}{Y} \right) (\boldsymbol{\sigma} - \boldsymbol{\alpha}) - \sqrt{\frac{a}{\bar{\alpha}_{*}}} \boldsymbol{\alpha}_{*} \right] \dot{p},$$

$$\dot{p} = \sqrt{(2/3)\boldsymbol{D}^{p}: \boldsymbol{D}^{p}}, \quad \bar{\alpha}_{*} = \phi(\boldsymbol{\alpha}_{*}), \quad a = B + R - Y,$$



Livermore Software Technology Corporation

Fig. 2. Schematic illustration of the two-surface mode

D I' C

Sounding surface changes in both its size and location:

$$\dot{\mathbf{R}} = k(R_{sat} - R)\dot{p},$$
$$\overset{\circ}{\mathbf{\beta}'} = k\left(\frac{2}{3}b\mathbf{D}^p - \mathbf{\beta}'\dot{p}\right)$$

$$\sigma_{bound} = B + R + \beta = B + (R_{sat} + b)(1 - e^{-k\epsilon^{p}}).$$



Work-hardening stagnation

$$g_{\sigma}(\boldsymbol{\sigma}',\boldsymbol{q}',r') = \frac{3}{2}(\boldsymbol{\sigma}'-\boldsymbol{q}'):(\boldsymbol{\sigma}'-\boldsymbol{q}')-r^2=0,$$

$$\vec{q}' = \mu(\beta' - q').$$

$$\vec{r} = h\Gamma, \quad \Gamma = \frac{3(\beta' - q'):\beta'}{2r}$$
(a) when $\dot{k} = 0$
(b) when $\dot{k} > 0$



Material Data Fitting Comparisons (DP600)



elsaiced parameters can nicely represent the stress-strain curves

Yoshida's Model Characteristics

Yoshida's Model can give good fit of the test

- The effective strain usually small (<0.16)</p>
- Yoshida Model shows saturation of stress





Chaboche's Model (M103)

- Difficulty in using it
 - There are eight user-defined material variables, make it difficult for ordinary user to use it
 - The build-in curve fitting only works for one stress-strain curve
 - Uniaxial-tension curve has to be used
 - It is impossible to get an reasonable material parameters
- An optimization algorithm has been developed
 - It is a stand-along code
 - Up to eight stress-strain curves can be used as input
 - The file names should begin from curve1.inc, curve2.inc...
 - Many iterations will be needed

 $\overline{\sigma} = \sigma_0 + Q_{r1}(1 - \exp(-C_{r1}\overline{\varepsilon})) + Q_{r2}(1 - \exp(-C_{r2}\overline{\varepsilon})) + Q_{x1}(1 - \exp(-C_{x1}\overline{\varepsilon})) + Q_{x2}(1 - \exp(-C_{x2}\overline{\varepsilon}))$



Livermore Software Technology Corporation

Chaboche' Mixed-Hardening Model DP600





Livermore Software Technology Corporation

Chaboche' Mixed-Hardening Model Saturation problem

Chaboche's Model has saturation problem

- The stress strain curve is extended by power law
- Then fit the curve and obtain the parameters
- The fitted curve does not match the experimental data



Modifications of Yoshida's Model New Proposal

A new algorithm has been proposed and tested

- Use a power-law to replace the R calculation in Yoshida's model
- The old function: $\dot{R} = k(R_{sat} R)\dot{p}$,
- The new function: R=Rsat(e+e0) ⁿ-Rsat(e0)ⁿ
- This new function will not have saturation problem
- There are two more parameters need to be fit: e0 and n



Modifications of Yoshida's Model Example: DP600





Modifications of Yoshida's Model Example: DP600



Livermore

Example: DP600



Livermore Sof

Some New Keywords

important for line-die simulation



Coordinate-based constraint

Why we need to put constraints on coordinates

- It can be more accurate to constraint the model in a fixture
- It can automate line-die simulation

The keyword is

- *CONSTRAINED_COORDINATE
- It can also be applied to local coordinate system
- Coordinates can be obtained from stationary tool in the CASE before



ID	IDPT	IDIR(I	DOF)	x	/ Z	CID			
unique ID,	PartI), DOI	F (one at	a time)					
1	1	3	1326.28	-100.236	156.434				
2	1	3	1276.21	159.983	138.517				
3	1	3	2466.03	-100.241	156.464				
4	1	3	2516.35	151.889	138.81				
5	1	1	2454.17	121.142	135.007				
6	1	2	2454.17	121.142	135.007				
7	1	2	1339.84	118.347	135.237				





Line Die Simulation

NUMISHEET Fender on Air with Multi-flanging – Flanging in Three Areas



Line Die Simulation





New Option for Trimming



Define a seed node on stationary tool (not blank) for trimming; use negative option



Livermore Software Technology Corporation

*node									
105226,2026.19,292.148,-134.788									
*DEI	*DEFINE CURVE TRIM NEW								
\$#	tcid	tctype	tflg	tdir	tctol	toln	nseed		
	1 2 0 0 0.250 -105226								
deck_trimline.iges									

Or,

*DE	*DEFINE_CURVE_TRIM_NEW								
\$#	tcid	tctype	tflg	tdir	tctol		toln	nseed	
	1	2		0	0.250				
dec	deck_trimline.iges								
*de	*define_trim_seed_point_coordinates								
\$	NSEED	X1	Y1	Z	1	X2	Y2	2	
Z2									

1 2026.19 292.148 -134.788

Trimming







Livermore Software Technology Corporation

Reflect light on a stamped panel





- Smoothed surface for stoning and curvature calculation
- Stoning direction can be manually defined by two nodes, or input # of directions without Node1 and Node2 definition

Stoning method – no mesh refinement 2.0mm element size around

- 2.0mm element size around door handle untrimmed – one way curvature.
- Mesh built with surface (not splitting from a coarser mesh)
- Results are in the order of 1.0E-04 to 1.0E-05.



3.000e-03

Summary:

Basically no differentiation in surface lows.

2 mm



*CONTROL FORMING STONING							\frown		
Ş	ISTONE	LENGTH	WIDTH	STEP	(DIRECT	REVERSE	METHOD		
	1	150.0	4.0	1.00	2	0	\ ♀		
Ş	NODE1	NODE2	SID	ITYPE					
.,	,,1,2								

Stoning method



Summary:

Expected.



Livermore Software Technology Corporation



1.800e-01

Surface Defect (Surface Low) Prediction 1.600e-01 Stoning method 1.440e-01 1.280e-01 1.1206-01 9.600e-02 8.000e-02 6.400e-02 4800e-02 3.200e-02 1.600e-02 0.000e+00_ Lower area of the handler moved down 0.02mm – one way curvature.

Summary:

Expected.



*C	ONTROL_FOR	MING_STONIN	IG		\frown	\frown
Ş	ISTONE	LENGTH	WIDTH	STEP	DIRECT REVERS	E METHOD
	1	300.0	4.0	0.10	(8)	0 (0)
Ş	NODE1	NODE2	SID	ITYPE		
Ş	stone orie	ntation in	all direct	ion		
.,	1,2					



Summary:

Expected.



*CONTROL FORMING STONING							\frown		
Ş	ISTONE	LENGTH	WIDTH	STEP	(DIRECT	REVERSE	METHOD		
	1	150.0	4.0	1.00	2	0			
Ş	NODE1	NODE2	SID	ITYPE					
.,	,,1,2								

Curvature method





Summary:

Captures the boundary of the surface low area. Inside of the boundary no curvature change. Expected.





Conclusions

- LSDYNA's Implicit capability becomes even more robust and efficient
- LSDYNA can continue maintains technical leader in sheet stamping simulation

